Accurate and Efficient Direct Numerical Simulation of drag reduction via riblets

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The drag reduction performance of the riblets depends on the sharpness of their tip.

Consequences for DNS: An extremely fine grid is required near the tip.

Adapted from Garcia–Mayoral & Jimenez, Phil. Trans. R. Soc. A (2011)

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Simulation of three dimensional riblets particularly expensive

Riblets resolved in immersed boundary solver

(Luchini, Eur. J. Mech. B Fluids (2016))

- **•** second-order finite differences on a staggered grid
- **•** implicit deferred correction of ∇ 2**u**
	- **•** solution behaves linearly at the wall

Riblets resolved in immersed boundary solver

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- **•** second-order finite differences on a staggered grid
- **•** implicit deferred correction of ∇ ²**u** and ∇p
	- **•** solution behaves linearly at the wall
	- **•** solution behaves as Stokes eigensolution at the riblet tip

Analytical solution of the Stokes problem around infinite corner:

> $\nabla \cdot \mathbf{u} = 0$ ✟✟ ∂*t* ✟✟✟ ✟✟ [∂]**^u** $+(u\cdot\nabla)u + \frac{1}{2}$ $\overline{\rho} - \nabla p = \nu \nabla^2 \mathbf{u}$

- **•** linear equations
- **•** two (**k** and **⊥**) uncoupled problems!

The problem k to the crest

1D Laplace problem

$$
0 = \nu \left(\frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)
$$

The problem ⊥ to the crest

2D Stokes problem

$$
\frac{\partial P}{\partial y} = \nu \left(\frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right)
$$

$$
\frac{\partial P}{\partial z} = \nu \left(\frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)
$$

- **•** For Stokes flows, the protrusion height $\Delta h = h_{\parallel} - h_{\perp}$ can be computed exactly
- **•** For turbulent flows, drag reduction performance is related to ∆*h*

Protrusion heights without and with corner correction with 8 (\bullet) and 16(\bullet) points per riblet (n):

Turbulent simulations: computational domain

We performed DNS of turbulent channel flows with both walls covred by riblets:

- Constant Pressure Gradient (CPG) $Re_τ = 200$
- **•** *s* ⁺ is changed by changing the phsical riblet size *s*/ δ

Example of riblet discretization

at
$$
s^+ = 8
$$
:

Straight riblets: Friction Coefficient

without and with corner correction with Grid 1 (\bullet) and Grid 2 (\bullet):

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STRAIGHT RIBLETS: FRICTION COEFFICIENT

without and with corner correction with Grid 1 (\bullet) and Grid 2 (\bullet):

Straight riblets: ∆*U* +

Comparison with literature data

Sinusoidal Riblets

Sinusoidal Riblets

Sinusoidal Riblets

Sinusoidal Riblets: Drag Reduction

Straight Long $(\lambda_x = 1500, \beta_{max} = 2^{\circ})$ Short ($\lambda_x = 250$, $\beta_{max} = 12^\circ$) ^y \boldsymbol{x}

An analytical correction for the tip singularity has been developed

- **•** accurate: increased accuracy in computing ∆*h*
- **•** efficient: accurate results with as low as 4 points per riblet

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and Outlook:

- **•** spanwise inhomogeneous grid, still some work to do...
- **•** we are interested in 3d riblets, but the result is general

Thank you for your kind attention!

complaints, comments, suggestions: davide.gatti@kit.edu

For comparison, Endrikat et al. (2021)

- **•** at least 32 points per riblet
- \bullet resolution for riblets at s⁺ = 10: δy⁺ = 0.057 ÷ 1.52, δz⁺ = 0.0334 ÷ 7.02
- **•** domain size 2.6δ **×** 0.64δ **×** 2δ

Our computational cost (finer grid)

• 91k core hours on bwUniCluster 2.0

Preliminary extension to 3D sinusoidal riblets

- **•** Global reference frame: decoupling into 1D Laplace and 2D Stokes problems fails
- **•** Local reference frame: decoupling is possible, but velocity components are intermixed
	- **•** discretization becomes explicit
	- **•** discretization becomes challenging due to staggered grid

Assumption: local misalignment of the riblets section is small $(\beta(x)_{max} = 2^{\circ}, \lambda_x^+$ x^+ = 1500)

$$
\begin{Bmatrix} u_G \\ v_G \end{Bmatrix} = \begin{bmatrix} f(\beta, c_{lap}, c_{st}) & g(\beta, c_{tap}, c_{st}) \\ p(\beta, c_{tap}, c_{st}) & q(\beta, c_{lap}, c_{st}) \end{bmatrix} \begin{Bmatrix} u_L \\ v_L \end{Bmatrix}
$$

Solution: limitation to the diagonal components of the correction matrix

3D riblets: preliminary results

Friction coefficient for the cases

- **•** smooth
- **•** with riblets
	- **•** without corner correction
	- **•** with corner correction

with

• 8 () • 16(\blacklozenge)

Analytical Corner Correction: Stokes problem with streamfunction-vorticity formulation

$$
\begin{cases} \nabla \cdot \mathbf{u} = 0 \\ \nabla^2 \mathbf{u} - \nu^{-1} \nabla p = 0 \end{cases} \implies \begin{cases} \nabla^2 \boldsymbol{\psi} = \boldsymbol{\omega} \\ \nabla^2 \boldsymbol{\omega} = 0. \end{cases}
$$

The steady $\psi - \omega$ Stokes system in polar coordinates is

$$
\frac{\partial^2 \psi}{\partial r^2} + \frac{1}{r} \frac{\partial \psi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \psi}{\partial \theta^2} = \omega
$$

$$
\frac{\partial^2 \omega}{\partial r^2} + \frac{1}{r} \frac{\partial \omega}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \omega}{\partial \theta^2} = 0.
$$

By imposing a variable separation for $\psi(r, \theta) = P(r)F(\theta)$ and $\omega(r, \theta) = R(r)G(\theta)$, c_9 imposing a variable separation is
calling $\chi = G''/G$ and $k = -\sqrt{\chi} < 0$:

$$
r^2R'' + rR' - \chi R = 0
$$

\n
$$
G'' + \chi G = 0 \implies R = ar^{-\sqrt{\chi}} + br^{\sqrt{\chi}} = ar^k
$$

since $r \ll 1$, we obtain:

$$
\omega(r,\theta)=r^k\left[C_1\cos\left(k\theta\right)+C_2\sin\left(k\theta\right)\right].
$$

 $\psi(r, \theta) = r^{k+2} \left[D_1 \cos\left((k+2) \theta \right) + D_2 \sin\left((k+2) \theta \right) + D_3 \cos\left(k \theta \right) + D_4 \sin\left(k \theta \right) \right]$.

The coefficients *Dⁱ* are given after the following boundary and symmetry conditions are provided:

The symmetry conditions lead to $D_2 = D_4 = 0$, and the definition of the stream-function gives u_r and u_θ depending on $\gamma = k + 1$ as

$$
u_r(r,\theta) = \frac{1}{r} \frac{\partial \psi}{\partial \theta} = -r^{\gamma} \left[D_1(\gamma + 1) \sin ((\gamma + 1) \theta) + D_3(\gamma - 1) \sin ((\gamma - 1) \theta) \right]
$$

$$
u_{\theta}(r,\theta) = -\frac{\partial \psi}{\partial r} = -(\gamma + 1)r^{\gamma} \left[D_1 \cos ((\gamma + 1) \theta) + D_3 \cos ((\gamma - 1) \theta) \right].
$$

The boundary conditions are used to find the ratio between the coefficients D_3 and D_1 , that is

$$
\frac{D_3}{D_1} = \frac{\cos\left(\left(\gamma + 1\right)\varphi_w\right)}{\cos\left(\left(\gamma - 1\right)\varphi_w\right)}.\tag{1}
$$

We set $D_1 = 1$.

The last constant to find is γ , whose value is given solving numerically $det(\mathbf{Q}(\gamma)) = 0.$

$$
\underbrace{\left[(\gamma + 1) \sin ((\gamma + 1) \varphi_w) \quad (\gamma - 1) \sin ((\gamma - 1) \varphi_w) \right] \left[D_1 \atop D_3 \right]}_{\mathbf{Q}(\gamma)} = \underbrace{\left[0 \atop \cos ((\gamma + 1) \varphi_w) \right]}_{\mathbf{Q}(\gamma)}
$$

The solution depends on the geometry considered: for the problem at hand, with $\varphi_w = \pi/6$, the result is $\gamma \approx 0.51222$.

The last unknown for the Stokes problem is the pressure:

$$
\nu \left[\frac{\partial^2 u_r}{\partial r^2} + \frac{1}{r} \frac{\partial u_r}{\partial r} + \frac{1}{r^2} \left(\frac{\partial^2 u_r}{\partial \theta^2} - 2 \frac{\partial u_\theta}{\partial \theta} - u_r \right) \right] - \frac{1}{\rho} \frac{\partial p}{\partial r} = 0
$$

$$
\nu \left[\frac{\partial^2 u_\theta}{\partial r^2} + \frac{1}{r} \frac{\partial u_\theta}{\partial r} + \frac{1}{r^2} \left(\frac{\partial^2 u_\theta}{\partial \theta^2} + 2 \frac{\partial u_r}{\partial \theta} - u_\theta \right) \right] - \frac{1}{\rho} \frac{\partial p}{\partial \theta} = 0
$$

$$
\frac{1}{\nu} p(r, \theta) = -4 \gamma D_3 r^{\gamma - 1} \sin ((\gamma - 1) \theta).
$$

The expression for *p* can not be used itself, because it is not guaranteed that *p* is symmetric and continuous inside the body. A correction can be implemented to choose a continuous branch for the solution, considering $\tilde{\theta} = \theta f(\theta)$ where $f(\theta) \neq 1$ only if $|\theta| > \varphi_w$, so that *p* is given by

$$
\frac{1}{\nu}p(r,\theta) = -4\gamma D_3 r^{\gamma-1} \sin((\gamma - 1)\theta f(\theta))
$$

$$
f(\theta) = \begin{cases} 1 + \frac{|\theta| - \pi}{\pi - \varphi_w} \left(\frac{1}{\gamma - 1} - 1\right) & \text{if } |\theta| > \varphi_w \\ 1 & \text{otherwise.} \end{cases}
$$

The Laplace problem reads:

$$
\nabla^2 u = 0 \Longrightarrow \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u}{\partial \theta^2} = 0,
$$

and a variable separation leads to the general solution

$$
u(r,\theta)=r^m\big[\mathcal{C}\cos(m\theta)+D\sin(m\theta)\big].
$$

No-slip boundary conditions, namely $u(r, \pm \varphi_w) = 0$, lead to cos $(m\varphi_w) = 0$ and so $m\varphi_w = \pi/2$. The symmetry condition, $u(r, \theta) = u(r, -\theta)$, gives $D = 0$ and the final expression for *u*, namely

$$
u=Cr^m\cos(m\theta).
$$

C here is a free constant that can be set to 1 to have a unique solution.

$$
u^{(t+\Delta t)} = u^{(t)} + (\text{lapl} + \text{NL} + \nabla p) \Delta t - u^{(t+\Delta t)} \text{imbc} \Delta t \Longrightarrow u^{(t+\Delta t)} = \frac{u^{(t)} + \text{RHS} \Delta t}{1 + \text{imbc} \Delta t}
$$

Being *uloc* and *ploc* the analytical solutions for the velocity and the pressure respectively, considering the problem for the *x*-direction one gets

$$
d_u = \underbrace{\left(\frac{lapl(u_{loc}(x,\cdot))}{Re} - \frac{p_{loc}(x+\Delta x,\cdot)-p_{loc}(x,\cdot)}{\Delta x}\right)}_{\text{corr}_{stokes}} u(x,\cdot),
$$

where *lapl*() is the laplacian corrected with the true distance from the body. The Navier-Stokes problem here is not so different: the terms to add inside *imbc* are a contribution from the Laplace problem in *u*, corr*lapl*, and from the Stokes problem in *v* and *w*, corr*stokes*.

Considering (u', v') in the local reference frame and (u, v) in the global one, the following additional rotation should be performed:

$$
u' = \cos(\beta) u + \sin(\beta) v, \qquad v' = \cos(\beta) v - \sin(\beta) u.
$$

The *imbc* coefficients in the local reference frame were already found for the straight riblets as

$$
d_{u'} = \text{corr}_{\text{lapl}} u', \qquad d_{v'} = \text{corr}_{\text{stokes}} v',
$$

but to define the corrections in the cartesian global reference frame the two components get mixed into the 2 **×** 2 non-diagonal system.

$$
\begin{bmatrix} d_u \\ d_v \end{bmatrix} = \begin{bmatrix} \cos^2{(\beta)} \text{corr}_{lapl} + \sin^2{(\beta)} \text{corr}_{stokes} & (\text{corr}_{lapl} - \text{corr}_{stokes}) \sin{(2\beta)/2} \\ (\text{corr}_{stokes} - \text{corr}_{lapl}) \sin{(2\beta)/2} & \cos^2{(\beta)} \text{corr}_{stokes} + \sin^2{(\beta)} \text{corr}_{lapl} \end{bmatrix} \begin{bmatrix} u \\ v \end{bmatrix}.
$$

$$
\begin{cases}\nd_u = (\cos^2(\beta) \text{corr}_{lapl} + \sin^2(\beta) \text{corr}_{stokes}) u \\
d_v = (\cos^2(\beta) \text{corr}_{stokes} + \sin^2(\beta) \text{corr}_{lapl}) v.\n\end{cases}
$$

Table 1: Results of the validation for straight riblets with the immersed boundary correction only (Standard) and with the addition of the corner correction (+ Correction). Errors are estimated as $(h-\bar{h})/\bar{h}$.

\bar{h} ^{*k*} ¯*h***[⊥]** ∆¯*h* 0.17150 0.08099 0.09051

Table 2: Protrusion heights reference values for *h*/*s* = **p** $\overline{3}/2.$

Table 3: U_h^+ $_b^+$ and C_f for the straight case. $\Delta\mathit{U}^+_b$ *b* and ∆*C^f* are evaluated considering the smooth channel simulation with the same δy^+ of the case considered.

Table 4: U_h^+ $_b^+$ and \emph{C}_{f} for the sinusoidal cases. $\Delta\emph{U}_{b}^+$ *b* and ∆*C^f* are evaluated considering the smooth channel simulation with the same δy^+ of the case considered.