

Turbulent drag reduction with streamwise travelling waves in the compressible regime

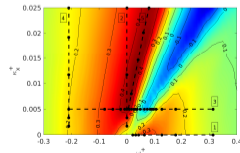
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18th European Turbulence Conference 2023

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Skin friction drag reduction with spanwise forcing

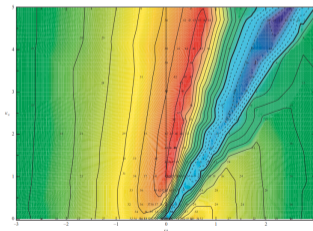
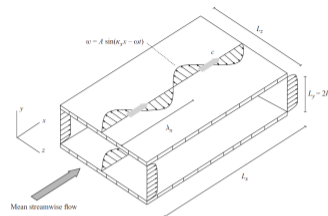
Streamwise-travelling waves of spanwise wall velocity

$$W(x, t) = A \sin(\kappa_x x - \omega t)$$

Oscillating wall: $\kappa_x = 0$

Steady wave: $\omega = 0$

$$\text{Drag reduction (DR)} = 1 - \frac{C_f}{C_{f,0}}$$

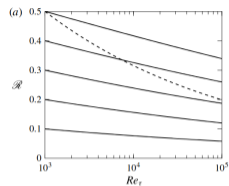


$Re_\tau = 200, A^+ = 12, DR_{max} = 48\%$
Quadrio et al. JFM 2009

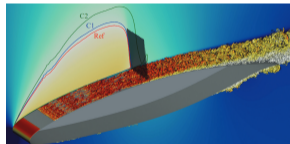
Towards real world applications

Dependence on:

- Reynolds number
- Complex shape and other drag sources
- Mach number



Gatti & Quadrio JFM 2016



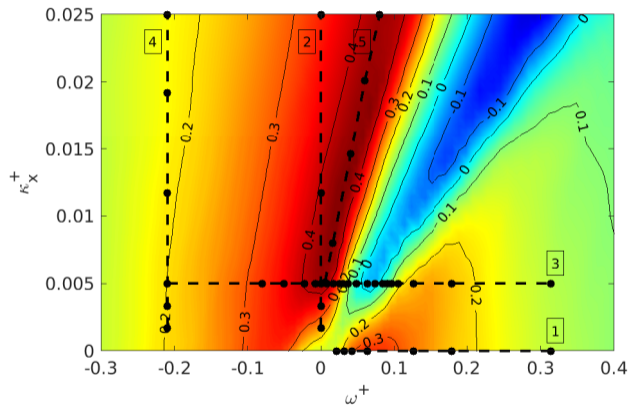
Quadrio et al. JFM 2022

≈ Yao & Hussain 2019

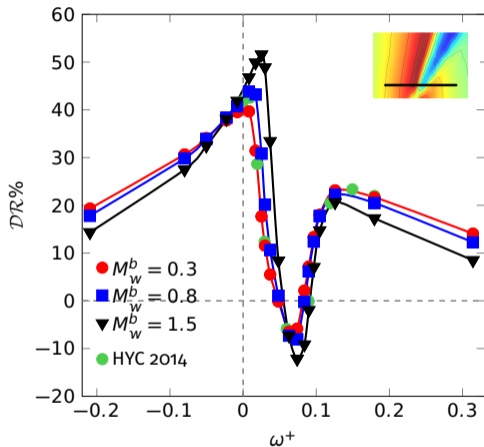
Simulation: parameters

- Direct Numerical Simulations
- Channel flow
- STREAMS solver (Bernardini et al. CPC 2021)
- Constant flow rate (CFR) : $U_b = \text{const}$
- $Re_\tau = 400$
- Parameters:
 - Mach: $M_w^b = U_b/c_w = U_b/\sqrt{\gamma RT_w} = 0.3, 0.8, 1.5$ (as Yao & Hussain JFM 2019)
 - Control: $A^+ = 12, 42$ combinations of (ω^+, κ_x^+)

Simulation: control map



Results: Travelling waves at $\kappa_x = 0.005$



$DR_{M=0.3} \approx 40\%$ $DR_{M=1.5} \approx 52\%$

Compressibility effect ($M_w^b \uparrow$):

- Negative effect at large $|\omega|$
- Negative effect in the drag increase zone
- Large positive effect at small ω

Question

- Travelling waves still work in compressible regime
- Compressibility has a positive effect on DR

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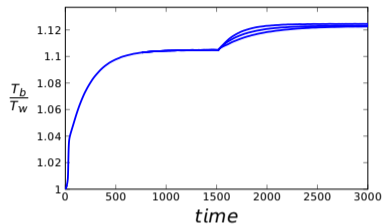
- Travelling waves still work in compressible regime
- Compressibility has a positive effect on DR

Is the increase of DR **actually** due to compressibility?

The effect of the temperature

- $T_b \uparrow$ with control depending on (ω, K_x)

$$T_b = \frac{1}{2h\rho_b U_b} \int_{-h}^h \langle \rho u T \rangle dy$$

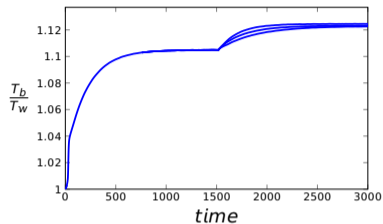


The effect of the temperature

- $T_b \uparrow$ with control depending on (ω, K_x)

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Compressibility or indirect $T_b \uparrow$?



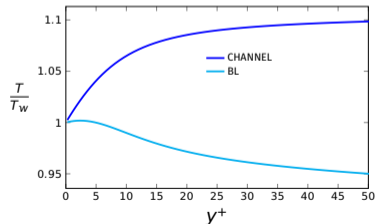
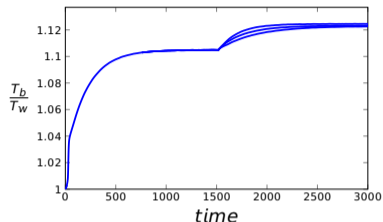
The effect of the temperature

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$$T_b = \frac{1}{2h\rho_b U_b} \int_{-h}^h \langle \rho u T \rangle dy$$

Compressibility or indirect $T_b \uparrow$?

- T profile not representative of external flows



New comparison strategy

$$\frac{\partial \rho e}{\partial t} + \frac{\partial \rho(e + p/\rho)u_j}{\partial x_j} = \frac{\partial \sigma_{ij}u_i}{\partial x_j} - \frac{\partial q_j}{\partial x_j} + fu_1$$

New comparison strategy

$$\frac{\partial \rho e}{\partial t} + \frac{\partial \rho(e + p/\rho)u_j}{\partial x_j} = \frac{\partial \sigma_{ij}u_i}{\partial x_j} - \frac{\partial q_j}{\partial x_j} + fu_1$$

Zero Bulk Cooling (ZBC)

New comparison strategy

$$\frac{\partial \rho e}{\partial t} + \frac{\partial \rho(e + p/\rho)u_j}{\partial x_j} = \frac{\partial \sigma_{ij}u_i}{\partial x_j} - \frac{\partial q_j}{\partial x_j} + fu_1 - \Phi$$

Zero Bulk Cooling (ZBC)

$$\Phi=0$$

Constrained Bulk Cooling (CBC)

$$\Phi=f(\Theta=\text{const})$$

New comparison strategy

$$\frac{\partial \rho e}{\partial t} + \frac{\partial \rho(e + p/\rho)u_j}{\partial x_j} = \frac{\partial \sigma_{ij}u_i}{\partial x_j} - \frac{\partial q_j}{\partial x_j} + fu_1 - \Phi$$

Zero Bulk Cooling (ZBC)

$$\Phi=0$$

Constrained Bulk Cooling (CBC)

$$\Phi=f(\Theta=\text{const})$$

Diabatic Parameter (Θ)

$$\Theta = \frac{T_w - T_b}{T_r - T_b}$$

$$T_r = \left(1 + \frac{\gamma-1}{2} r(M_w^b)^2\right) T_b$$

Θ = Fraction of the available kinetic energy transformed into thermal energy at the wall

New comparison strategy

We **decouple** compressibility from purely thermodynamical effects

Zero Bulk Cooling (**ZBC**)

Θ varies

↓

T_b varies

Constained Bulk Cooling (**CBC**)

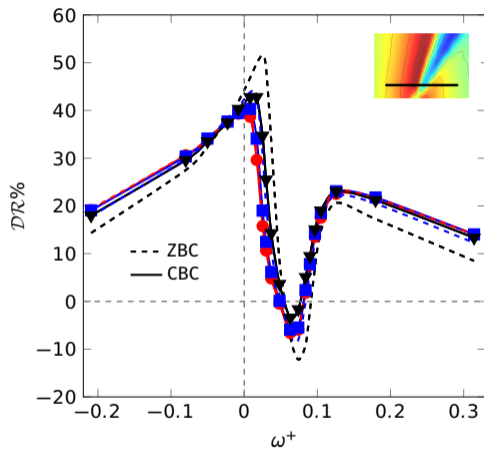
Θ constant = 0.75

↓

T_b

- **constant** across **control**
- **varies** across **Mach**

Drag reduction: ZBC vs CBC



$$DR_{M=0.3} \approx 40 \% \quad DR_{M=1.5}^{ZBC} \approx 52 \% \quad DR_{M=1.5}^{CBC} \approx 43 \%$$

Conclusions

- Influence of **compressibility** to the **drag reduction** performance of the streamwise-**travelling waves** of spanwise wall motion
- **Two** different **comparison strategies**:
 - Zero Bulk Cooling: Θ =variable
 - Constrained Bulk Cooling: Θ =const to **decouple** compressibility from purely thermodynamic effects
- Travelling waves still work in the compressible regime
- Compressibility effectiveness:
 - ZBC: large improvement
 - CBC: **marginal improvement**

Extension: Power budget

Drag reduction (\mathcal{DR})

$$\mathcal{DR} = 1 - \frac{C_f}{C_{f,0}}.$$

Input/control power (P_{in})

$$P_{in} = \frac{1}{P_0^*} \frac{1}{T_{ave} L_x L_z} \int_{t_i}^{t_f} \int_0^{L_x} \int_0^{L_z} W \tau_z dx dz dt$$

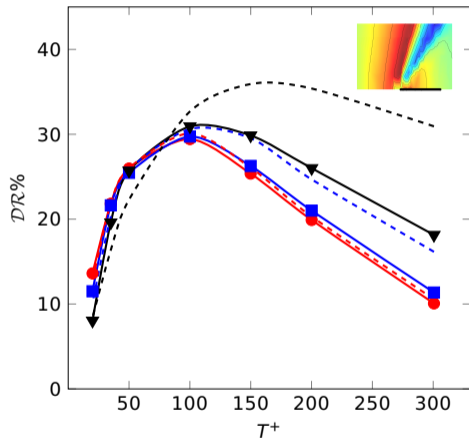
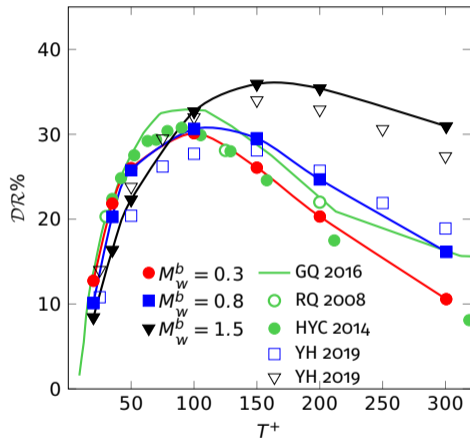
Pumping Power

$$P^* = \frac{U_b}{T_{ave} L_x L_z} \int_{t_i}^{t_f} \int_0^{L_x} \int_0^{L_z} \tau_x dx dz dt$$

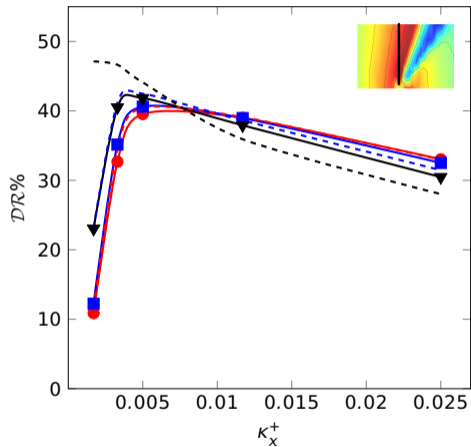
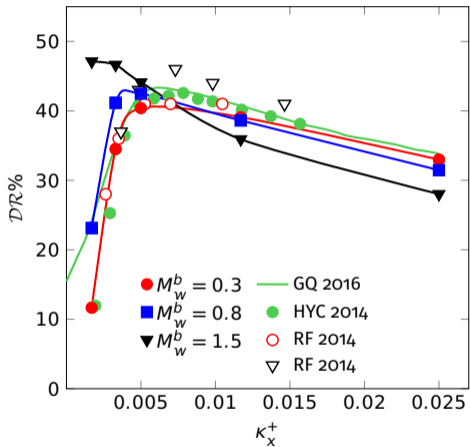
Net energy saving rate (P_{net})

$$P_{net} = \mathcal{DR} - P_{in}.$$

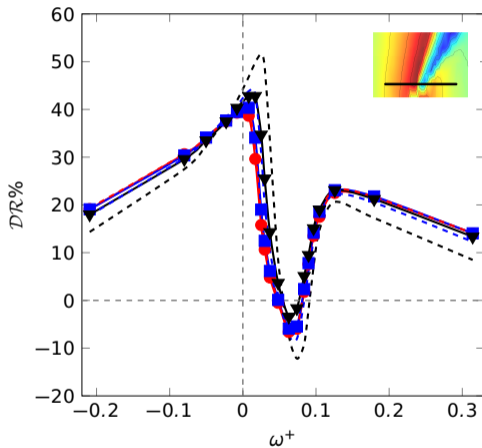
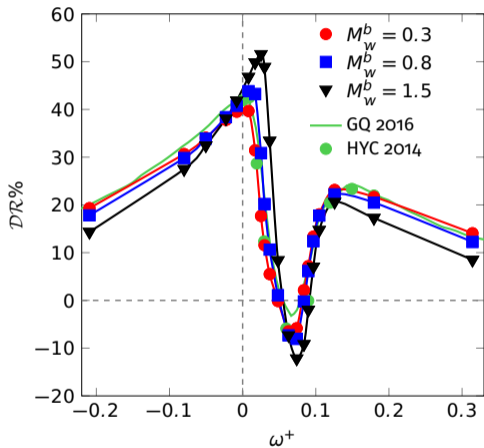
Line 1



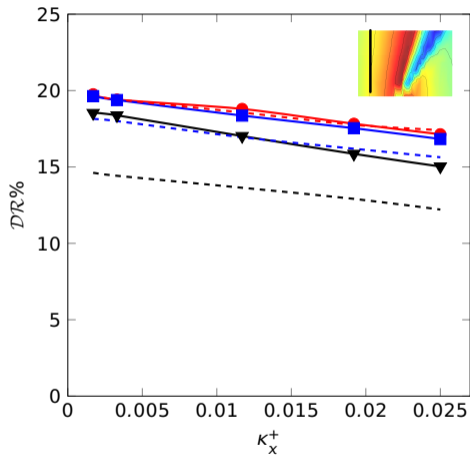
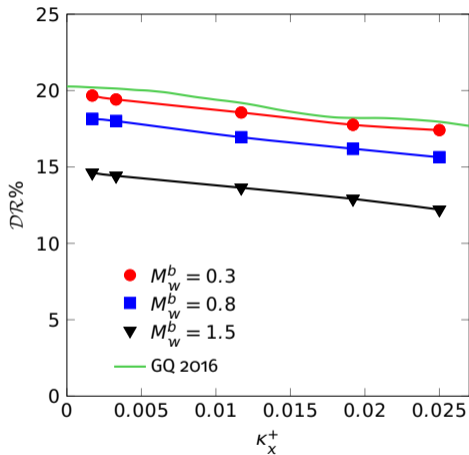
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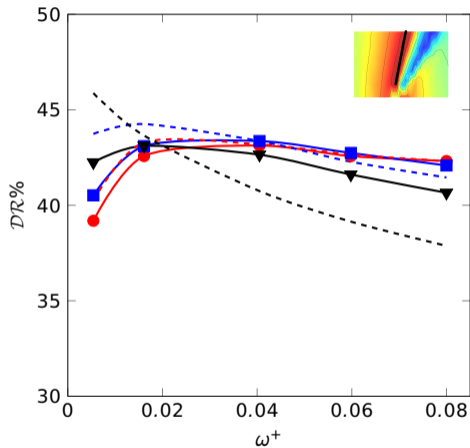
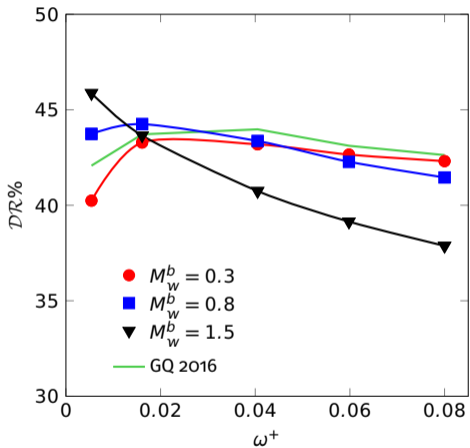
Line 3



Line 4

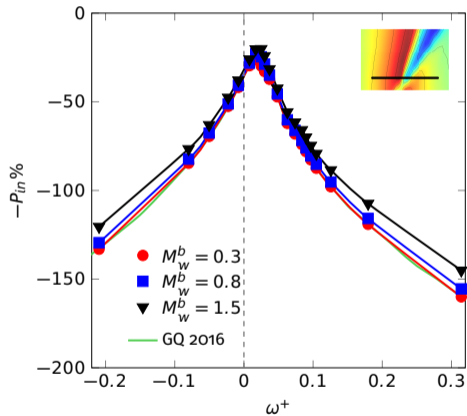


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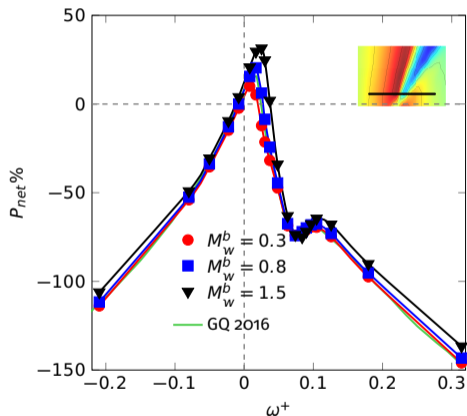


Results: Power budgets

Input/control power



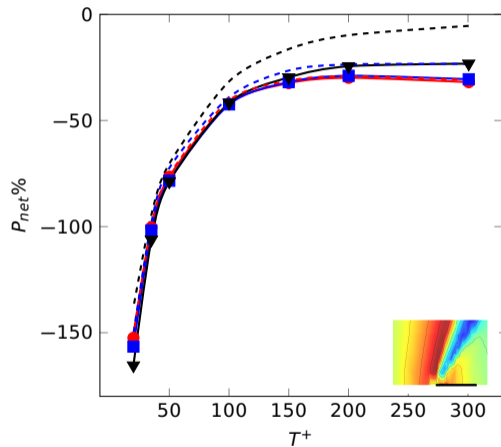
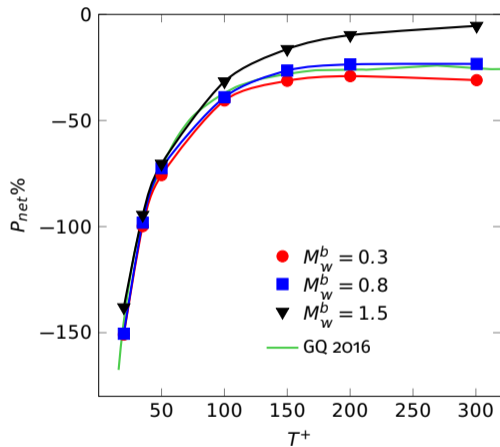
Net saving



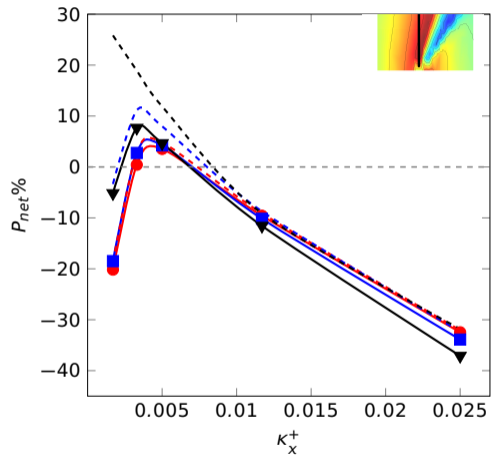
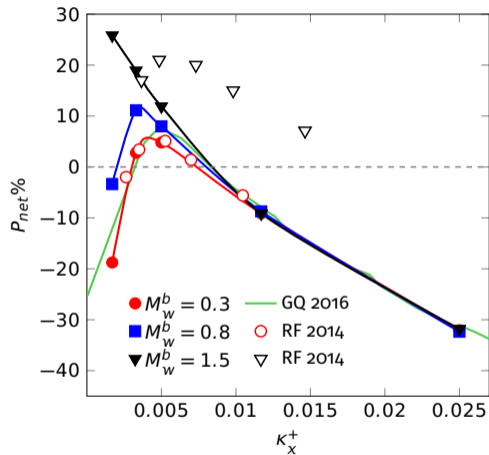
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$$P_{net} = DR - P_{in}$$

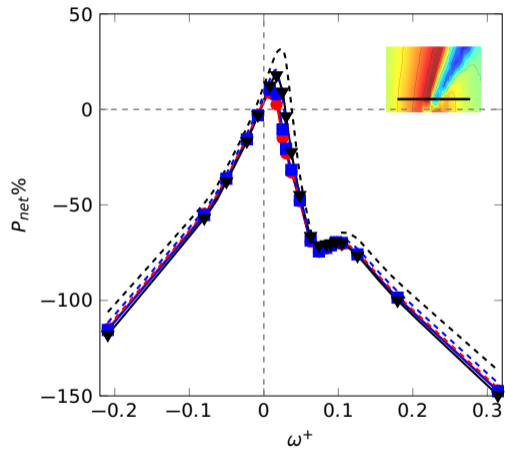
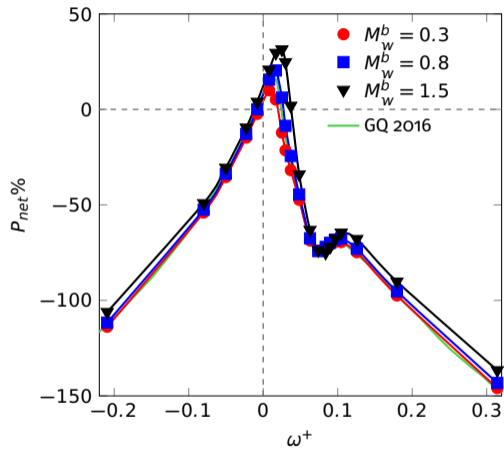
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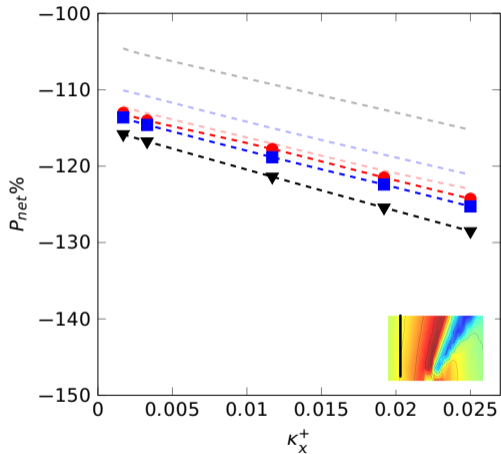
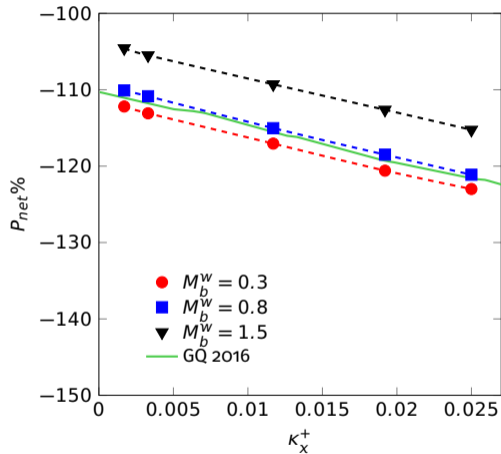
Line 2



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Line 4



Line 5

