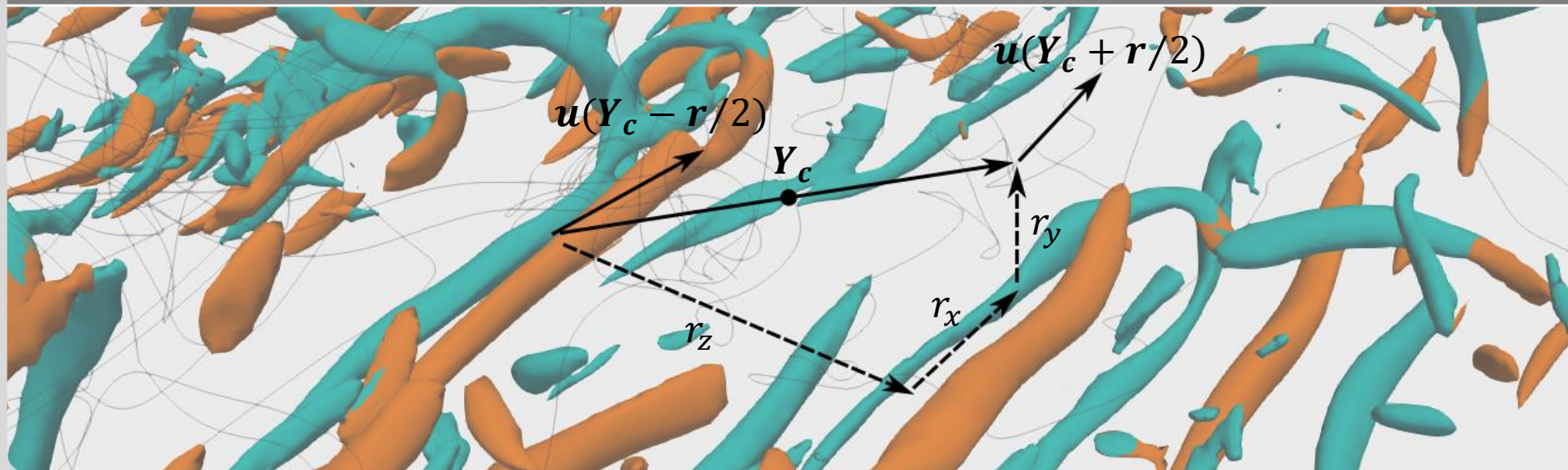


Scale energy fluxes in turbulent channels with drag reduction at constant power input

Daive Gatti, A. Remigi, A. Cimarelli,
Y. Hasegawa, B. Frohnepfel, M. Quadrio

16th EUROPEAN TURBULENCE CONFERENCE, Stockholm, Sweden



The question in drag reduction

“how do turbulent flows
with and without drag reduction
differ?”

- Comparison between different flows required
- Very different outcomes depending upon how flow is driven

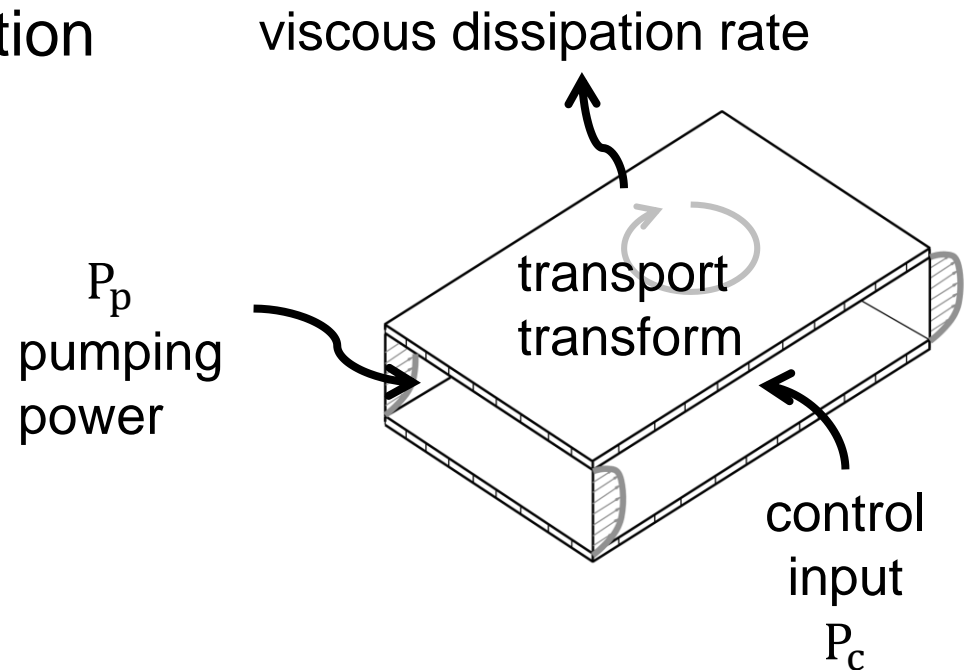
Our question in drag reduction

“how do turbulent flows
with and without drag reduction
differ **energetically**?”

- Comparison between different flows required
- Very different outcomes depending upon how flow is driven
- An example: **enstrophy** in drag reduced flows:
 - Ricco *et al.*, JFM12: at CPG **increases**
 - Agostini, *et al.*, JFM14: at CFR **decreases**
 - Gatti *et al.*, submitted: ...it depends!

Today's goal

“how do turbulent flows
with and without drag reduction
differ **energetically**?”

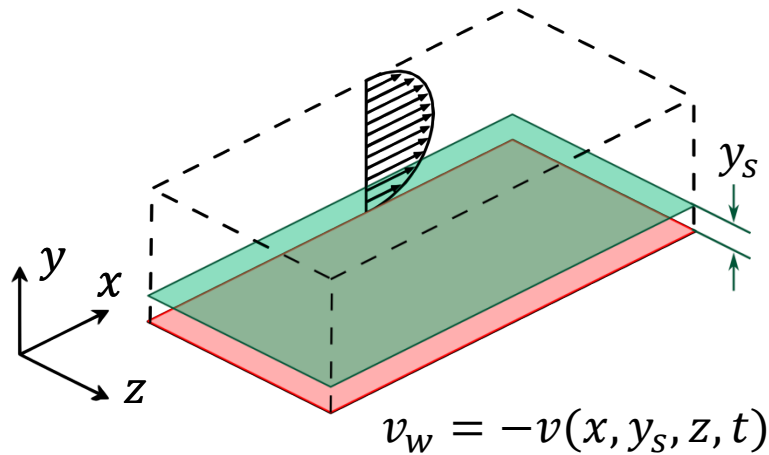


Assess **changes of scale energy** fluxes
in turbulent channels

driven **at Constant Power Input** (Hasegawa et al., JFM14)

A model control strategy

Opposition Control (Choi, Moin & Kim, JFM94)



reference

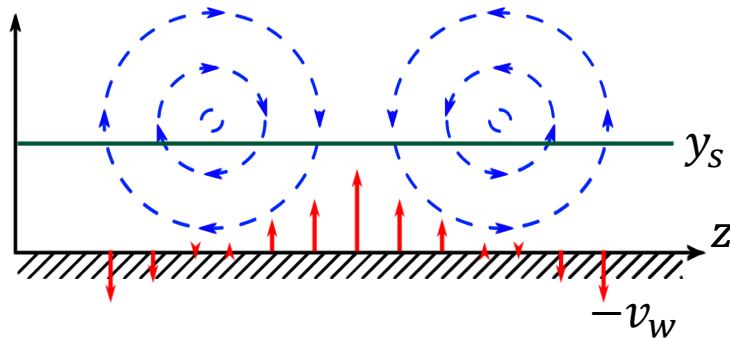
$$Re_\tau = \frac{u_\tau h}{\nu} = 200$$

$$Re_b = \frac{U_b h}{\nu} = 3177$$

controlled

$$Re_\tau = 190.5$$

$$Re_b = 3474$$



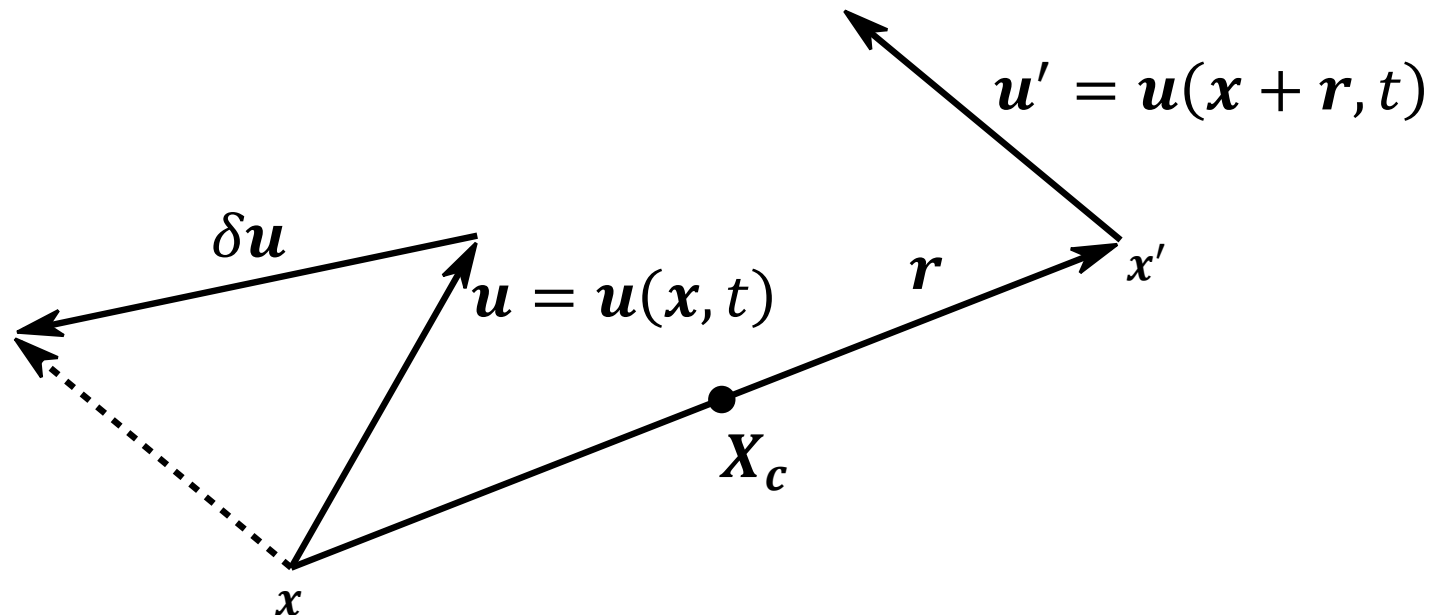
drag reduction $R = 1 - \frac{C_f}{C_{f,0}} = 23.9\%$

control power fraction $\gamma = \frac{P_c}{P_t} = 0.0035$

$$\frac{U_b}{U_{b,ref}} = 1.094$$

Second-order structure function (1)

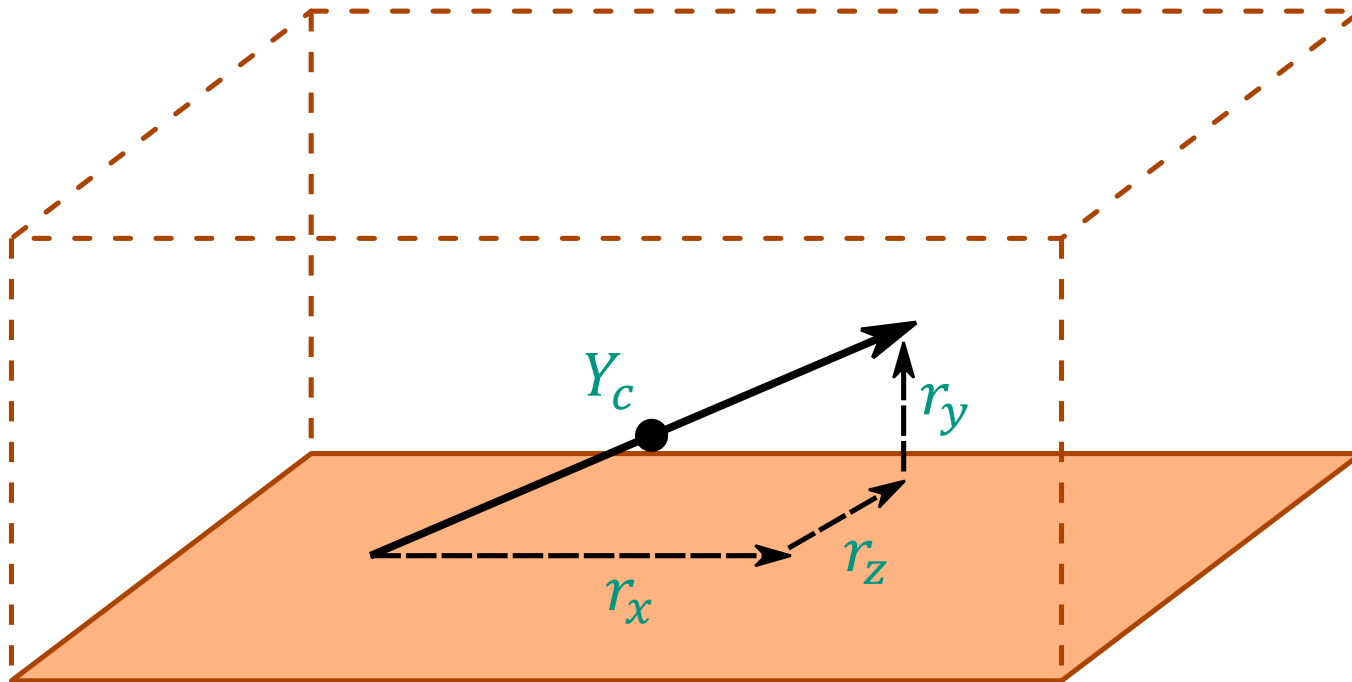
$$\langle \delta u^2 \rangle(\mathbf{r}, \mathbf{X}_c) = \langle [u(\mathbf{X}_c - \mathbf{r}/2) - u(\mathbf{X}_c + \mathbf{r}/2)]_i^2 \rangle$$



loosely speaking, amount of fluctuation **energy** at **scale** $\|\mathbf{r}\|$

Second-order structure function (2)

$$\langle \delta u^2 \rangle(\mathbf{r}, Y_c) = \langle [u(\mathbf{X}_c - \mathbf{r}/2) - u(\mathbf{X}_c + \mathbf{r}/2)]_i^2 \rangle$$

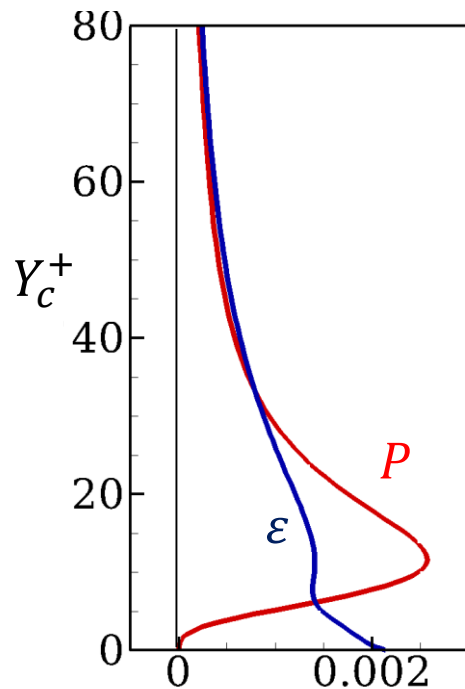


in channels function of wall-normal coordinate Y_c and vector \mathbf{r}

Kinetic energy budget

$$k = \frac{1}{2} \langle u_i^2 \rangle \text{ budget}$$

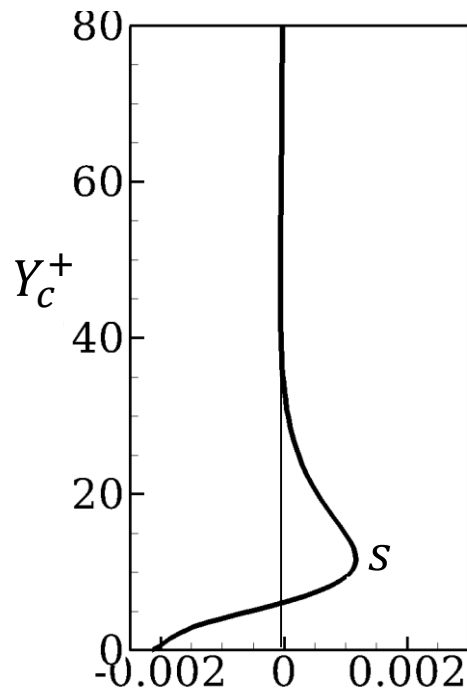
$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$



Kinetic energy budget

$$k = \frac{1}{2} \langle u_i^2 \rangle \text{ budget}$$

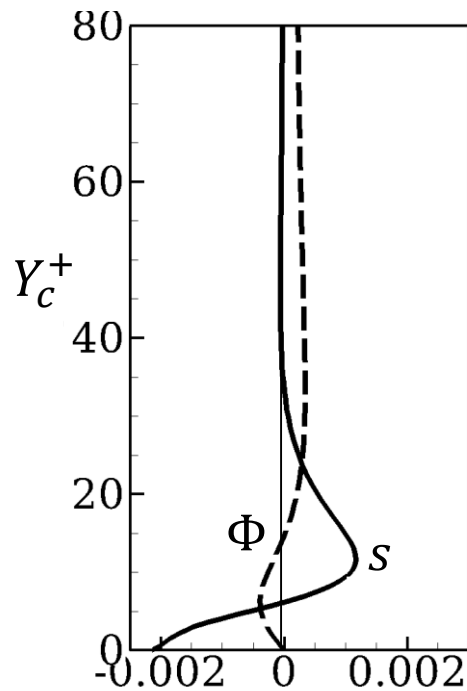
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Kinetic energy budget

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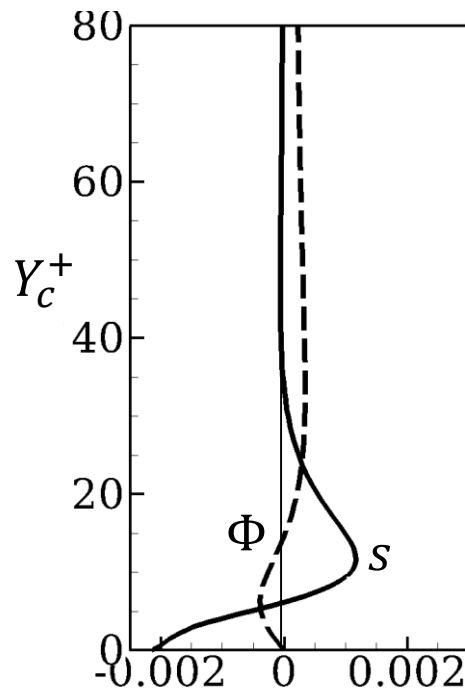
Scale energy budget

$k = \frac{1}{2}\langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$

$\langle \delta u^2 \rangle$ budget

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$



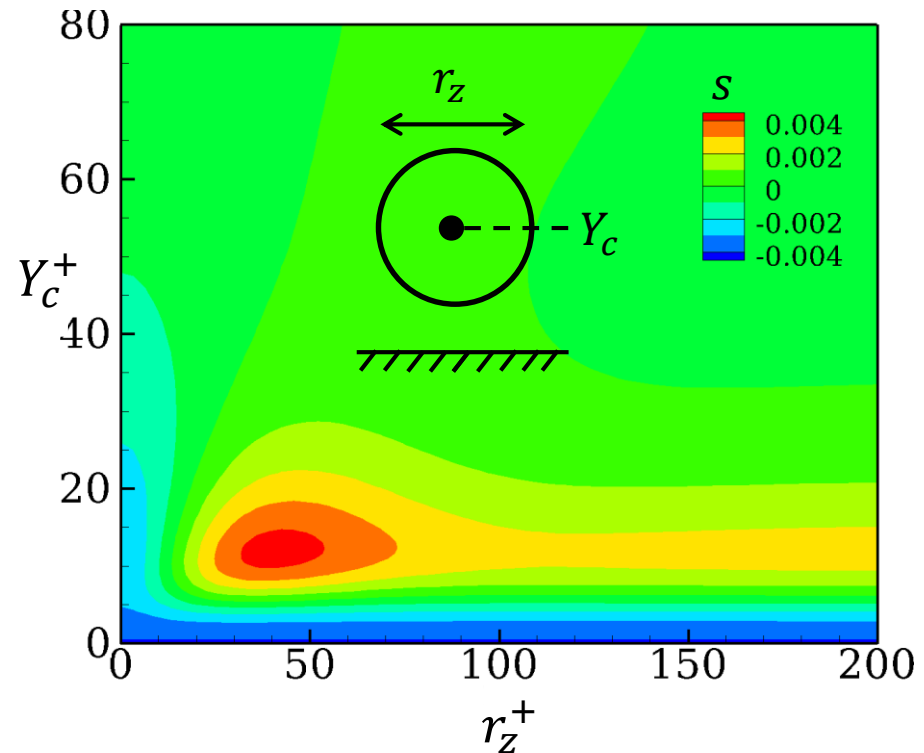
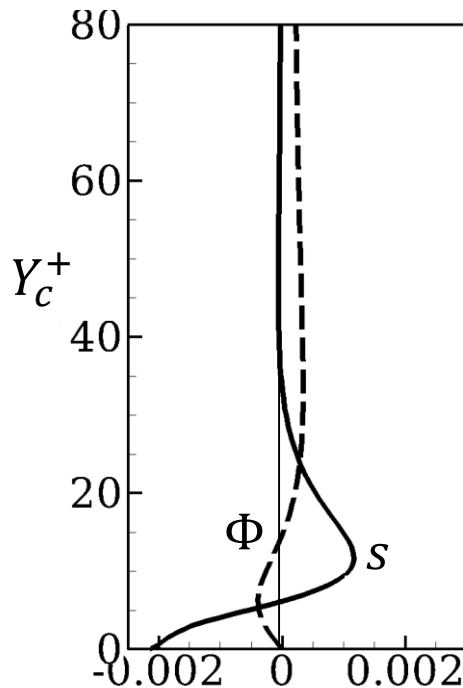
Scale energy budget: r_z, Y_c space

$k = \frac{1}{2}\langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$

$\langle \delta u^2 \rangle$ budget

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$



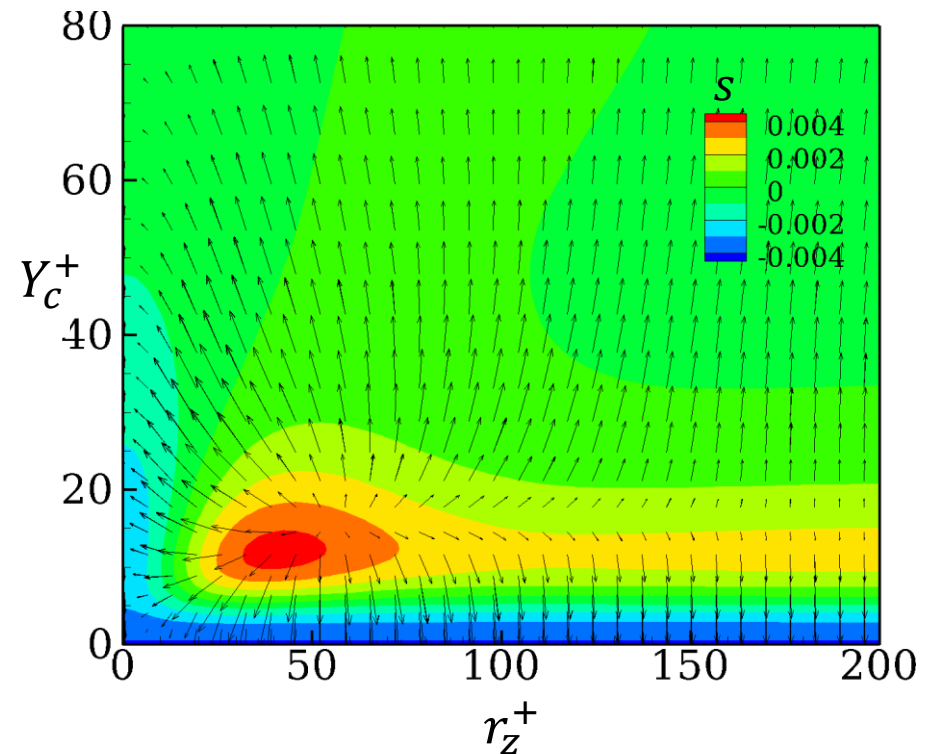
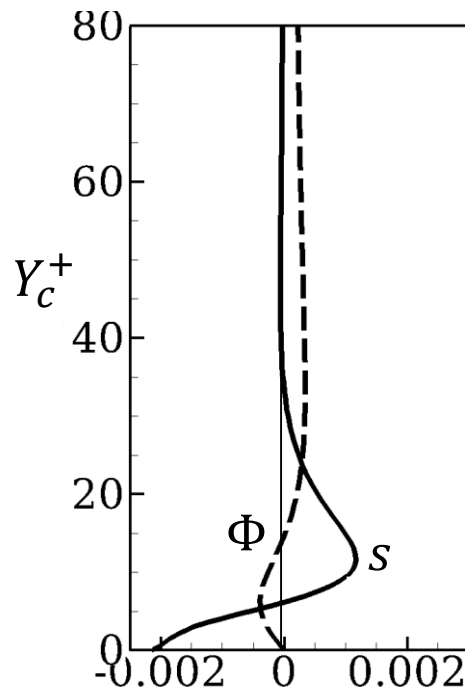
Scale energy budget: r_z, Y_c space

$k = \frac{1}{2} \langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$

$\langle \delta u^2 \rangle$ budget

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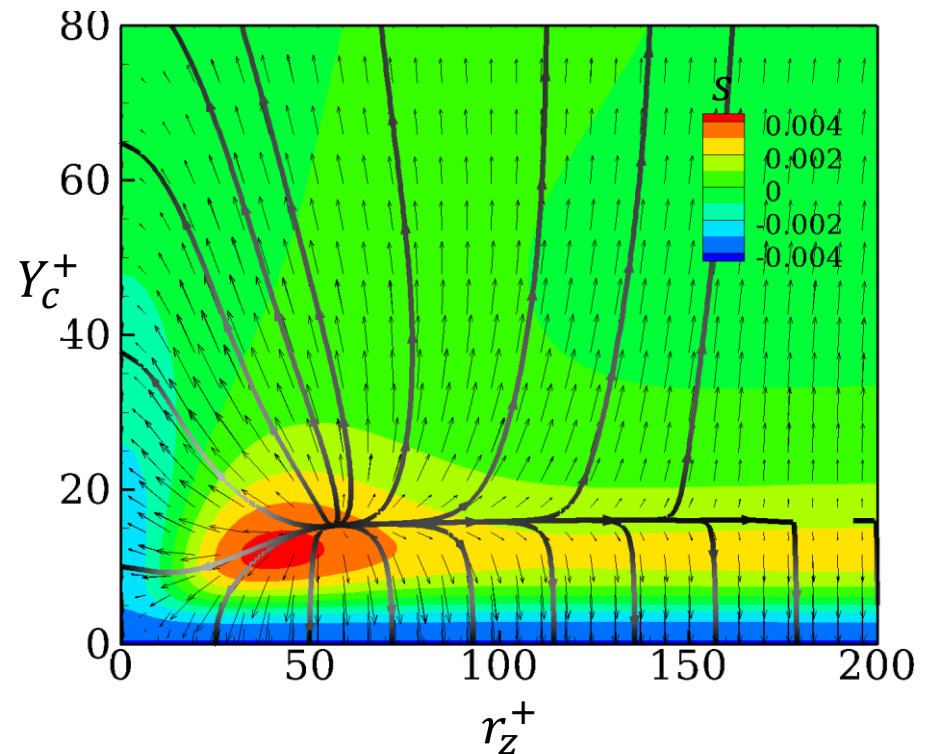
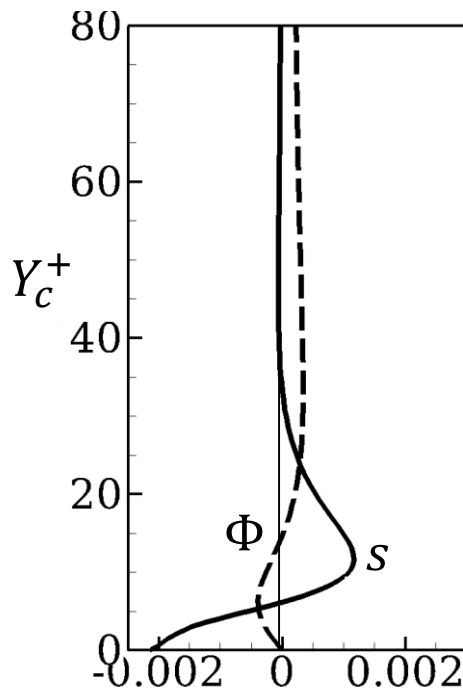
Scale energy budget: r_z, Y_c space

$k = \frac{1}{2} \langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$

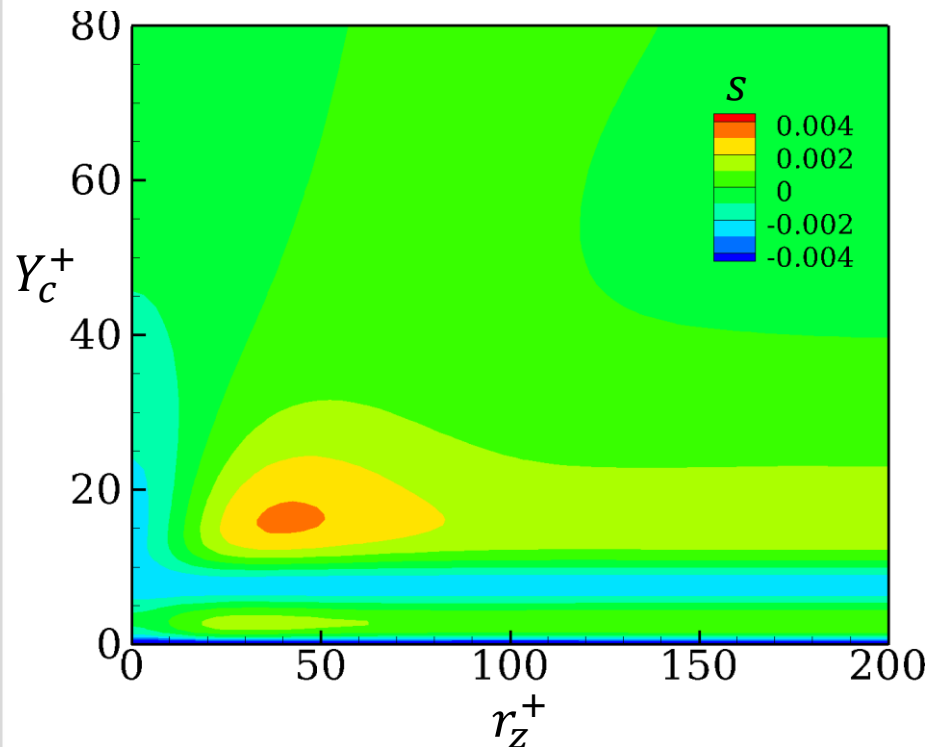
$\langle \delta u^2 \rangle$ budget

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$

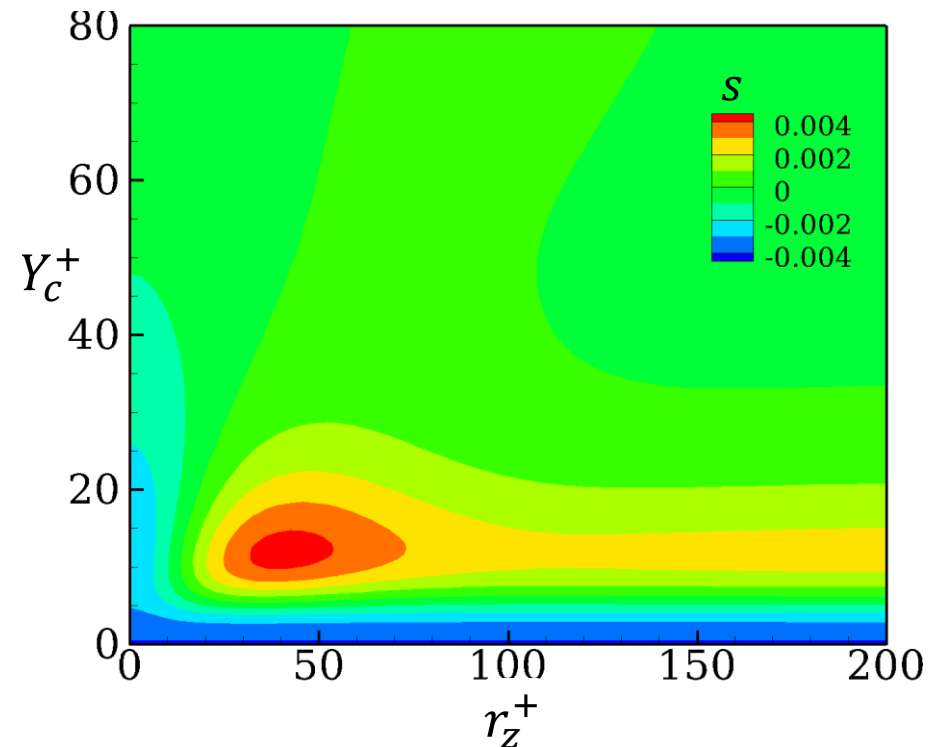


r_z, Y_c space with drag reduction

Opposition Control

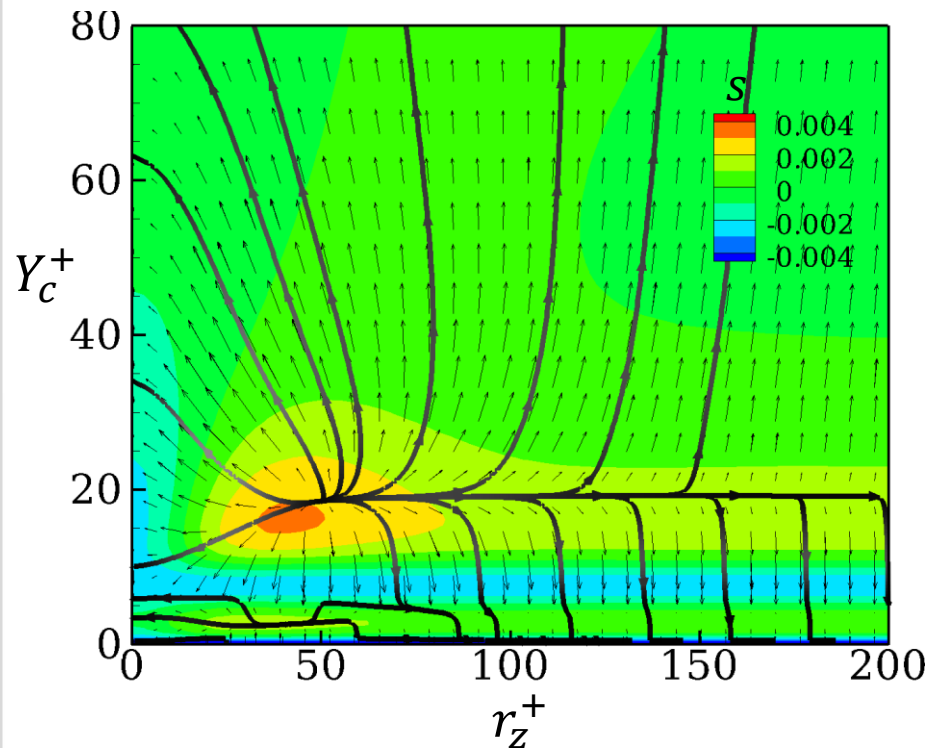


Reference

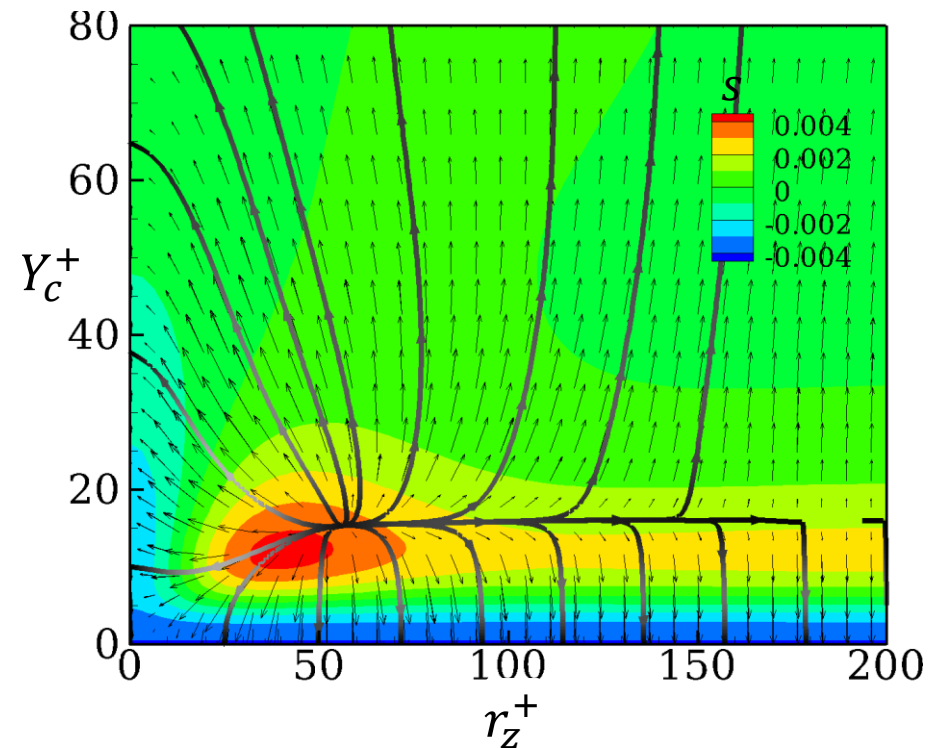


r_z, Y_c space with drag reduction

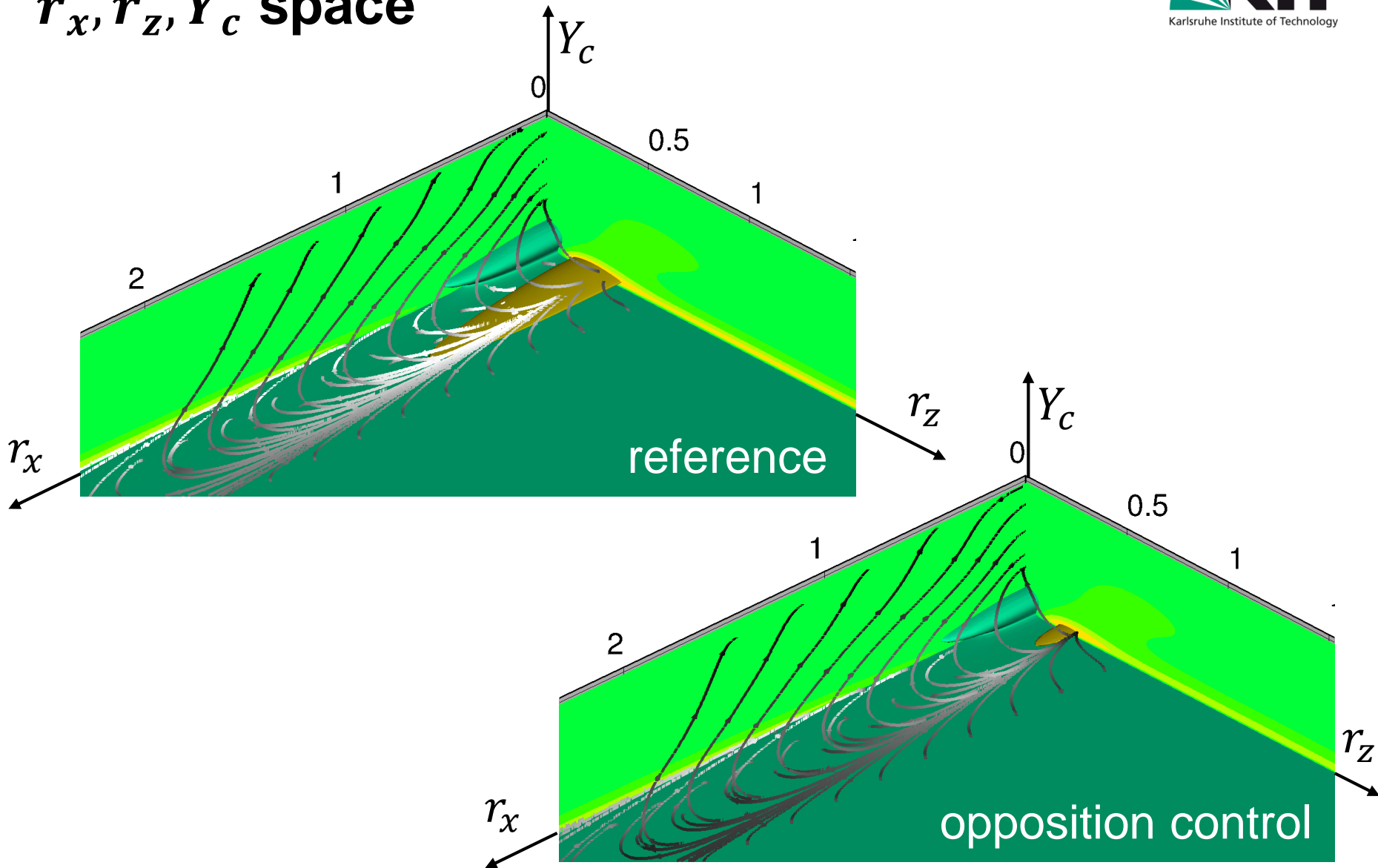
Opposition Control



Reference



r_x, r_z, Y_c space



Conclusion

- Constant Power Input (CPI) approach
 - is essential to assess energy transfer rates in drag-reduced flows
- Scale energy budget
 - is modified by the control in the near-wall region
 - highlights some mechanisms of drag reduction
- Paths of energy
 - only **small differences** in drag reduced flow...
 - ... when the comparison is fair! (CPI)
 - small differences **are important!!**

Outlook

- Quantitatively assess small control-induced changes
- Consider the whole 4D (r_x, r_y, r_z, Y_c) -space
- Consider the budget equation for $\langle \delta u \delta v \rangle$



THANKS
for your kind attention!

for questions, suggestions, complaints:

davide.gatti@kit.edu

Comparing energy transfer rates correctly

successful control $R = 1 - \frac{C_f}{C_{f,0}} > 0$ in turbulent channels

	U_b	$-\frac{dp}{dx}$	$P_p = -\frac{dp}{dx} hU_b$	$P_t = P_p + P_c$	C_f
CPG	↑	=	↑	↑	↓
CFR	=	↓	↓	?	↓

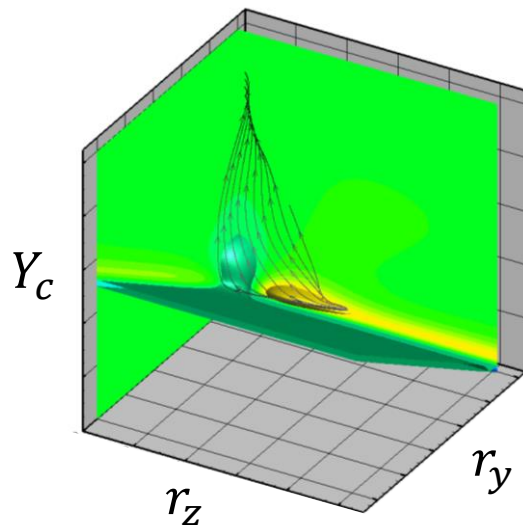
P_p and P_t change between controlled and natural flow!!

Hasegawa et al., JFM (2014) propose Constant Power Input:

CPI	↑	↓	↓	=	↓
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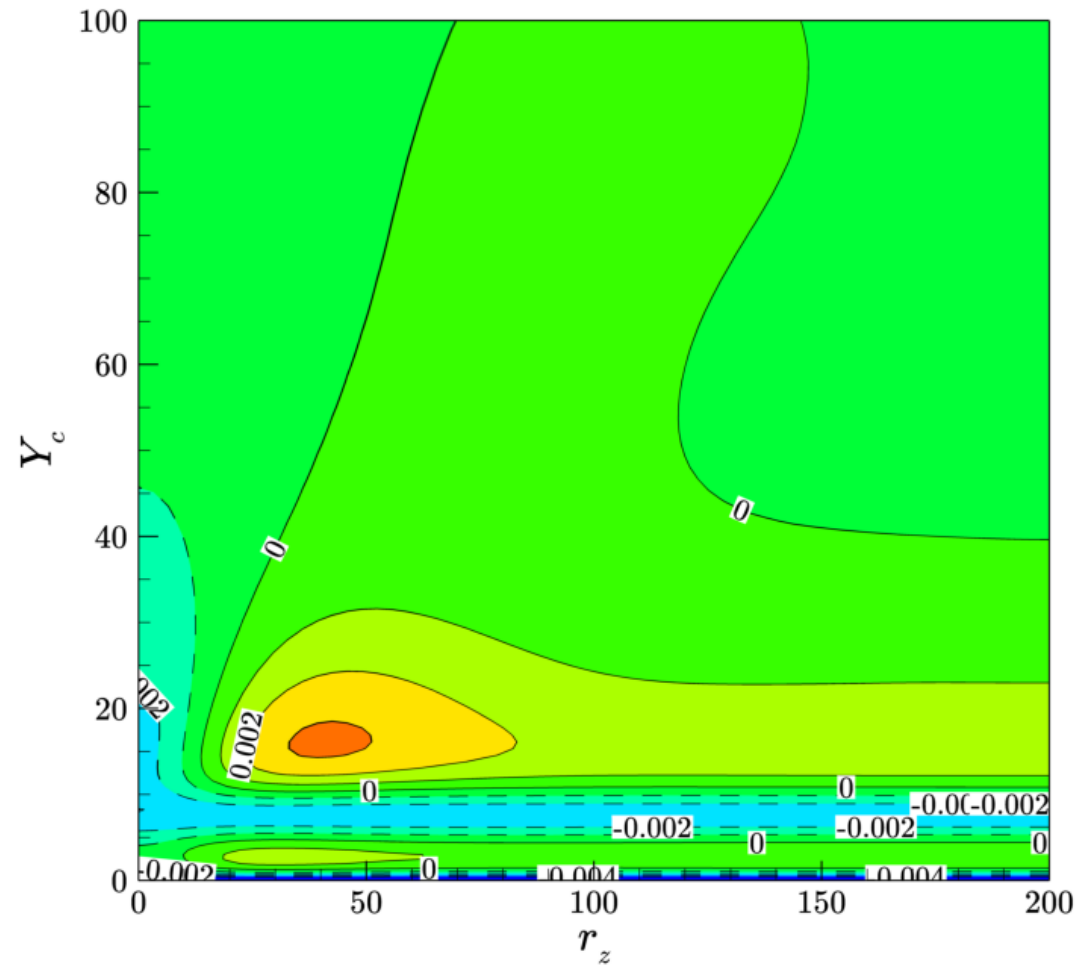
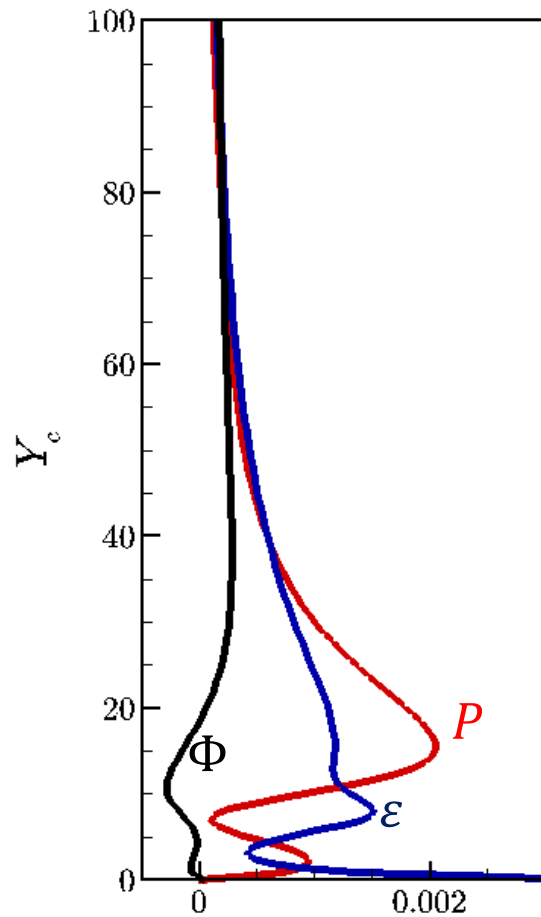
Outlook

- Quantitatively assess small control-induced changes
- Consider the whole 4D (r_x, r_y, r_z, Y_c) -space

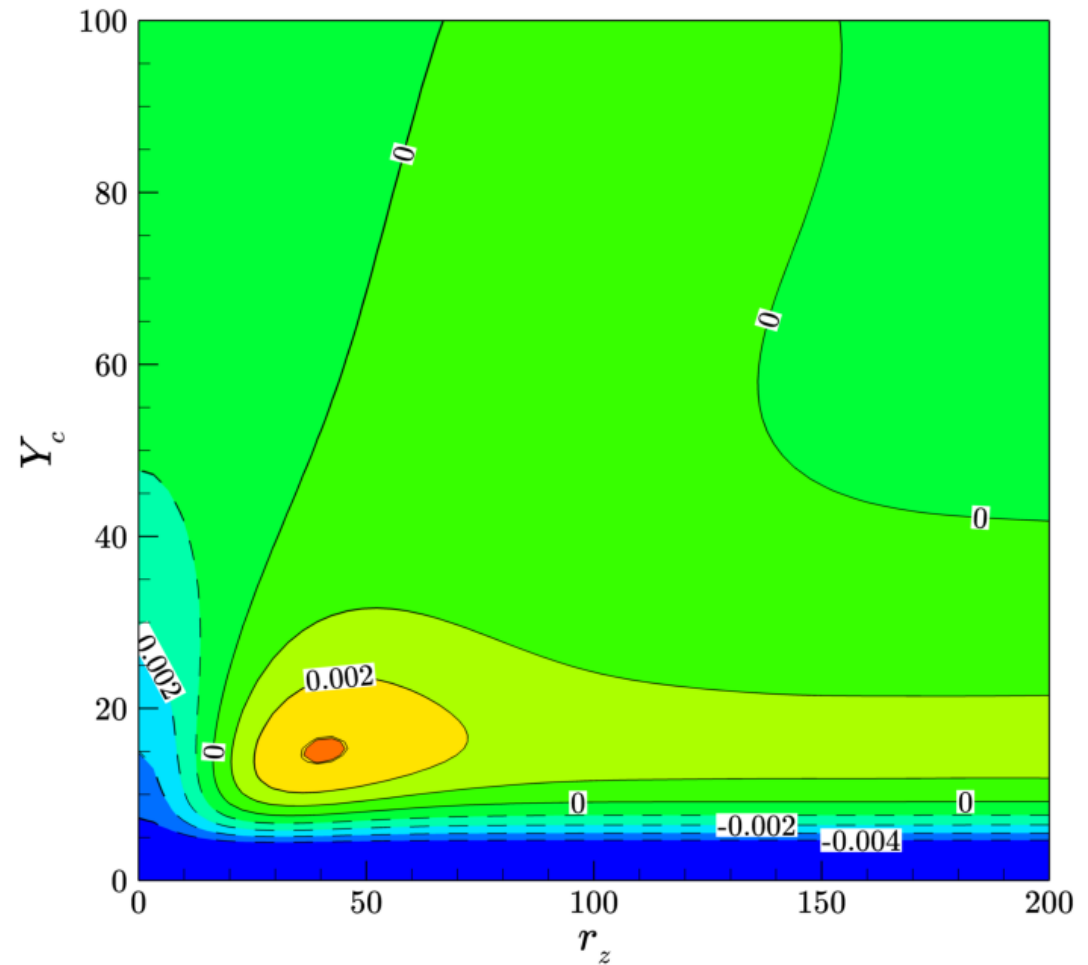
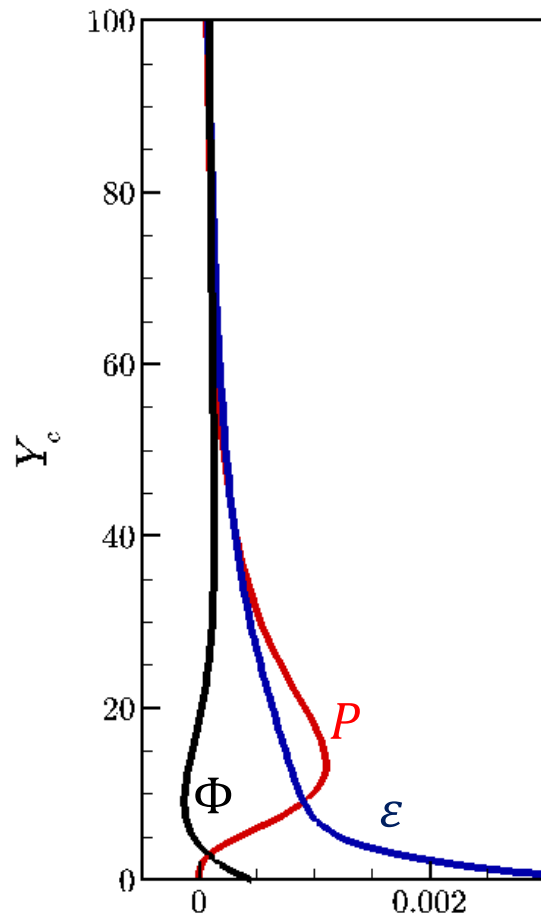


- Consider the budget equation for $\langle \delta u \delta v \rangle$

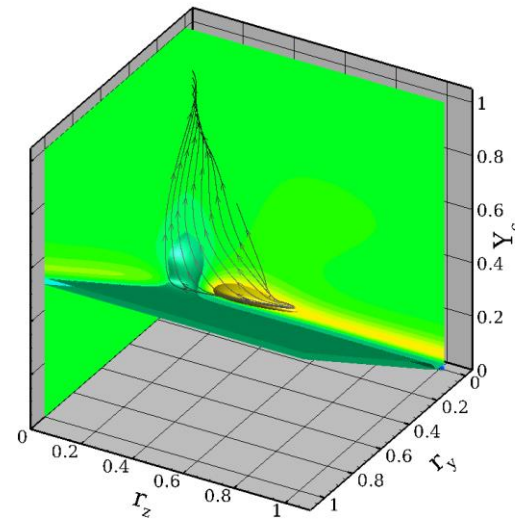
Results - vc



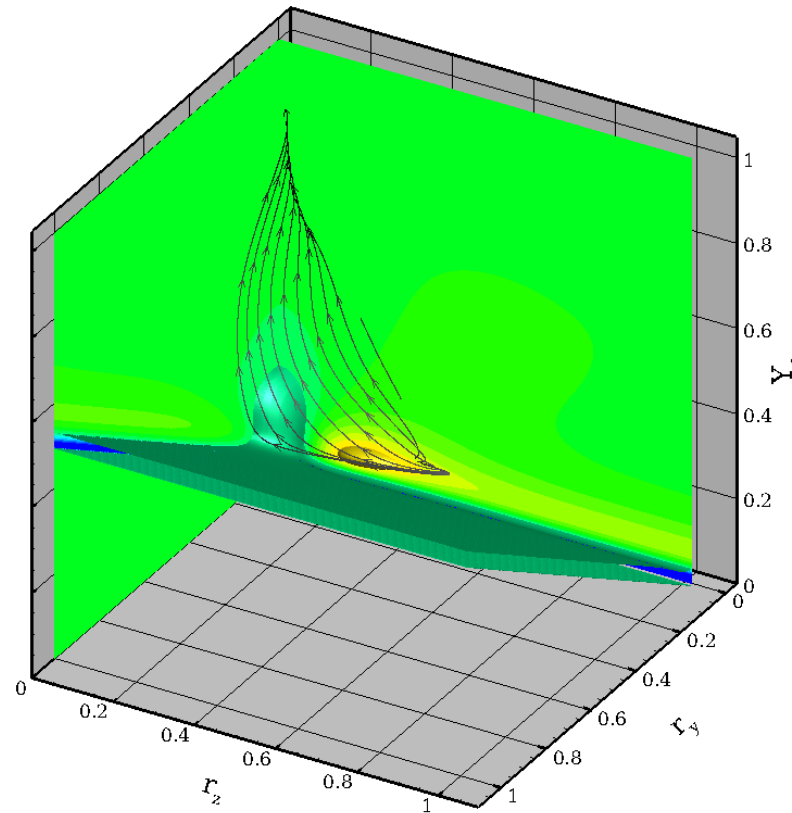
Results - ow



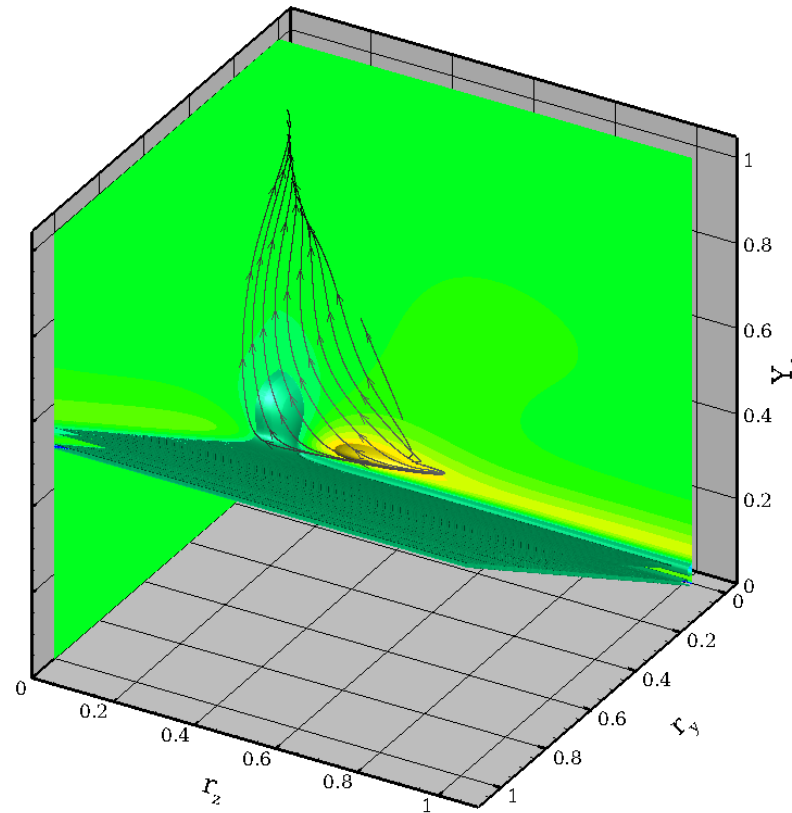
Results - ref



Results - ref



Results - ref



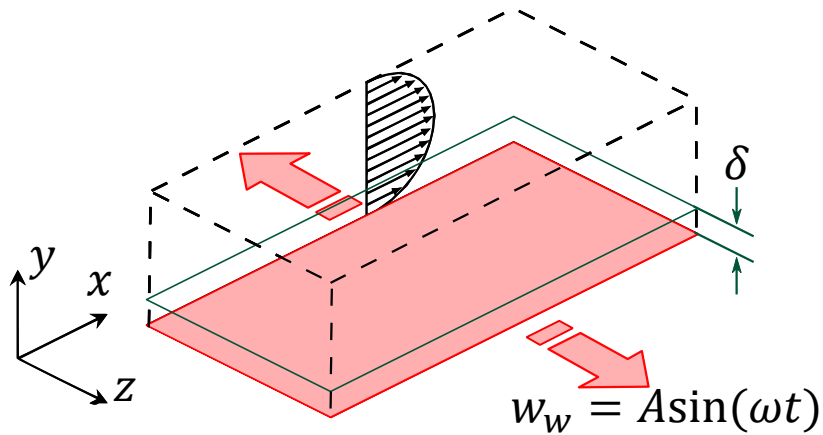
How to drive the flow?

successful control $R = 1 - \frac{C_f}{C_{f,0}} > 0$

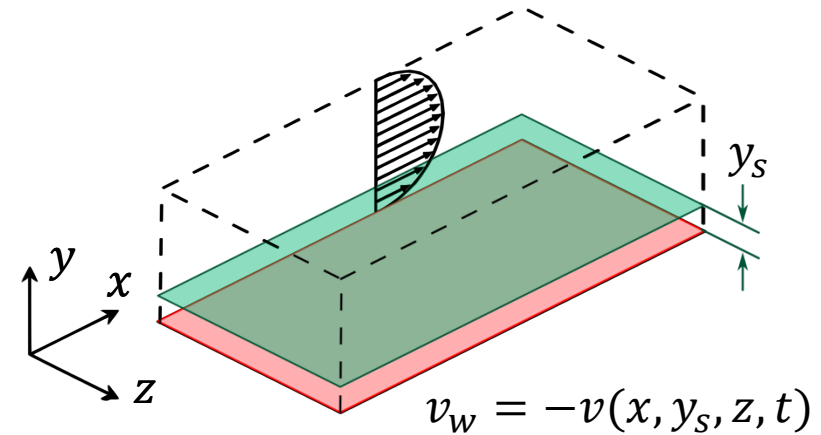
	U_b	$-\frac{dp}{dx}$	$P_p = -\frac{dp}{dx} h U_b$	$P_t = P_p + P_c$	C_f
CPG	↑	=	↑	↑	↓
CFR	=	↓	↓	?	↓

Control strategies

Spanwise wall oscillations



Opposition control



drag reduction $R = 1 - \frac{C_f}{C_{f,0}} = 17.1\%$

control power fraction $\gamma = \frac{P_c}{P_t} = 0.098$

$$\frac{U_b}{U_{b,ref}} = 1.028$$

$$R = 23.9\%$$

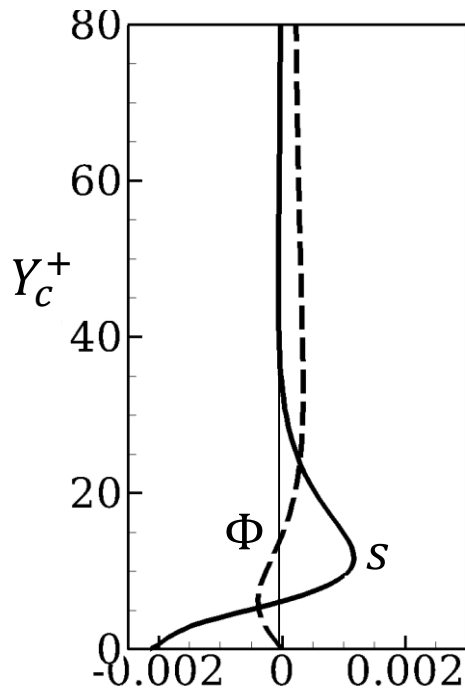
$$\gamma = 0.0035$$

$$\frac{U_b}{U_{b,ref}} = 1.094$$

Scale energy budget

$$k = \frac{1}{2} \langle u_i^2 \rangle \text{ budget}$$

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$



$$\langle \delta u^2 \rangle \text{ budget}$$

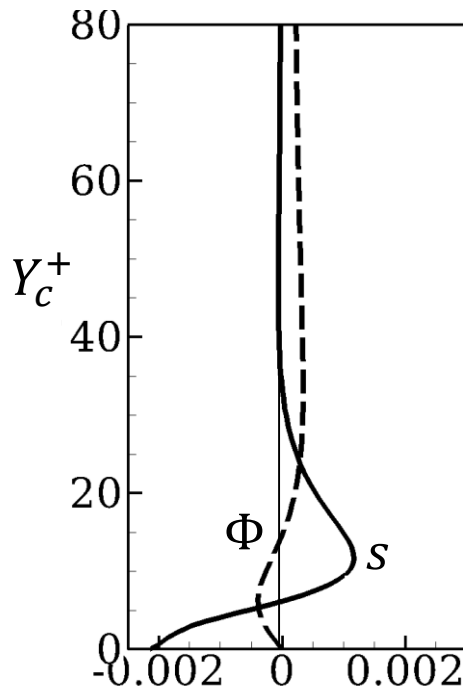
$$\underbrace{\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c}}_{\text{divergence of fluxes}} = s(\mathbf{r}, Y_c)$$

divergence of fluxes

Scale energy budget

$$k = \frac{1}{2} \langle u_i^2 \rangle \text{ budget}$$

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$



$$\langle \delta u^2 \rangle \text{ budget}$$

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$

↓
source term

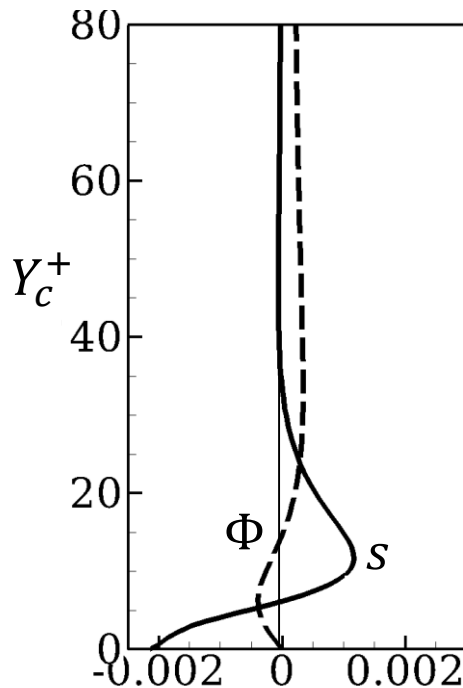
↓
 $s(\mathbf{r}, Y_c)$

budget between P and dissipation ϵ

Scale energy budget

$k = \frac{1}{2}\langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$



$\langle \delta u^2 \rangle$ budget

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$

↓
space flux

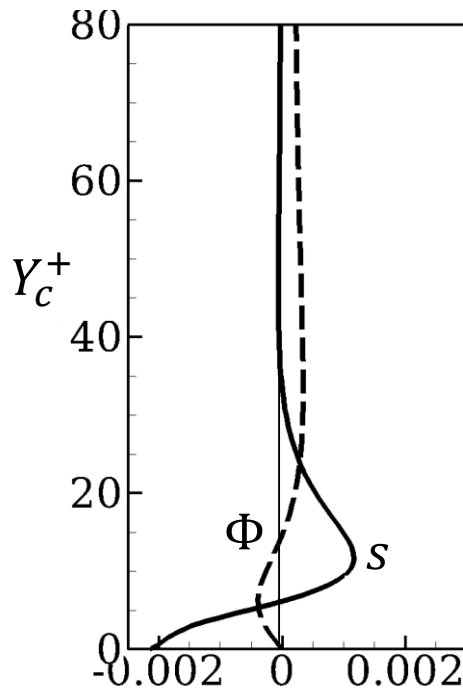
↓
 $\Phi_c(\mathbf{r}, Y_c)$

transport of $\langle \delta u^2 \rangle$ in geometric space
in a channel flow, transfer of energy
at scale \mathbf{r} in Y_c -direction

Scale energy budget

$k = \frac{1}{2}\langle u_i^2 \rangle$ budget

$$\frac{d\Phi}{dY_c} = P - \epsilon = s(Y_c)$$



$\langle \delta u^2 \rangle$ budget

$$\nabla_r \cdot \Phi_r + \frac{\partial \Phi_c}{\partial Y_c} = s(\mathbf{r}, Y_c)$$

↓
scale flux



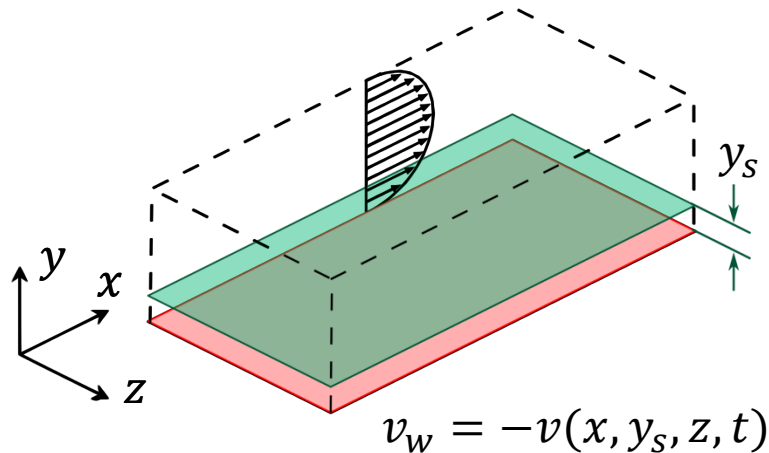
$$\Phi_r(\mathbf{r}, Y_c)$$

transport of $\langle \delta u^2 \rangle$ across scales r

not visible in the TKE budget!

A model control strategy

Opposition Control (Choi, Moin, Kim, JFM94)



reference

$$Re_{\Pi} = \frac{U_{\pi} h}{\nu} = 6500$$

$$Re_{\tau} = \frac{u_{\tau} h}{\nu} = 200$$

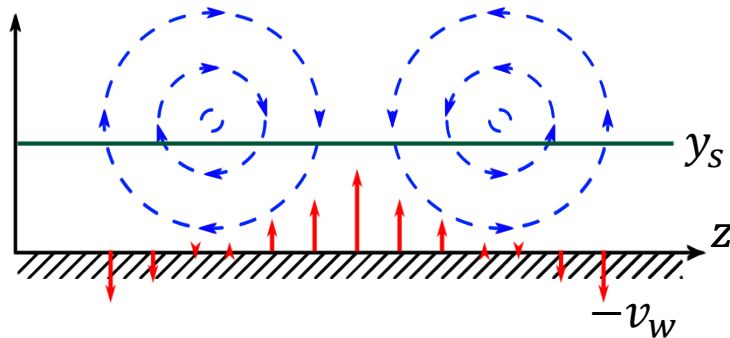
$$Re_b = \frac{U_b h}{\nu} = 3177$$

controlled

$$Re_{\Pi} = 6500$$

$$Re_{\tau} = 190.5$$

$$Re_b = 3474$$



drag reduction $R = 1 - \frac{C_f}{C_{f,0}} = 23.9\%$

control power fraction $\gamma = \frac{P_c}{P_t} = 0.0035$

$$\frac{U_b}{U_{b,ref}} = 1.094$$