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## Direct Numerical Simulation of Drag Reduction with Uniform Blowing over a Rough Wall

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## Background

#### Turbulence

- Huge drag
- Environmental problems
- High operation cost
- How to control?

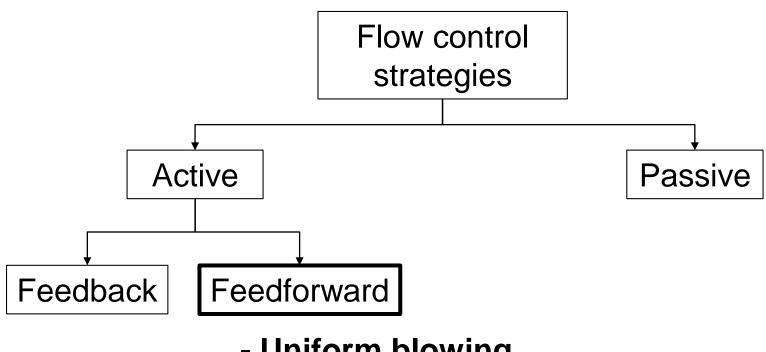






## **Flow control classification**

(M. Gad-el-Hak, J. Aircraft, 2001)



- Uniform blowing

## **Uniform blowing (UB)**

(Sumitani & Kasagi, *AIAA J.*, 1995 Kametani & Fukagata, *J. Fluid Mech.*, 2011)

Drag contribution in a channel flow with UB(/US)

$$C_{f} = \frac{12}{\text{Re}_{b}} + 12 \int_{0}^{2} (1 - y)(-\overline{u'v'}) dy - 12V_{w} \int_{0}^{2} (1 - y)\overline{u} dy$$

$$V_{w}: \text{Blowing velocity}$$

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$$C_{0}(1 - y)\overline{u} dy$$

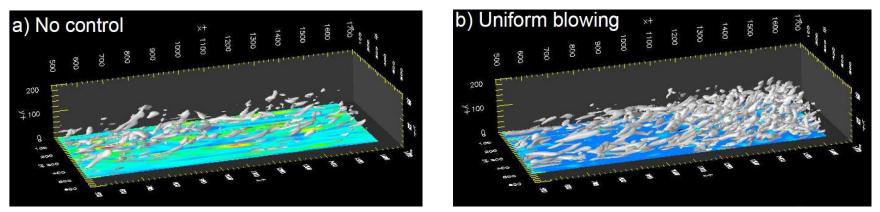
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$$C_{0}(1 - y)\overline{u} dy$$

$$C_{0}(1 - y)$$

- Excellent performance (about 45% by  $V_w = 0.5\% U_{\infty}$ )
- Unknown over a rough wall



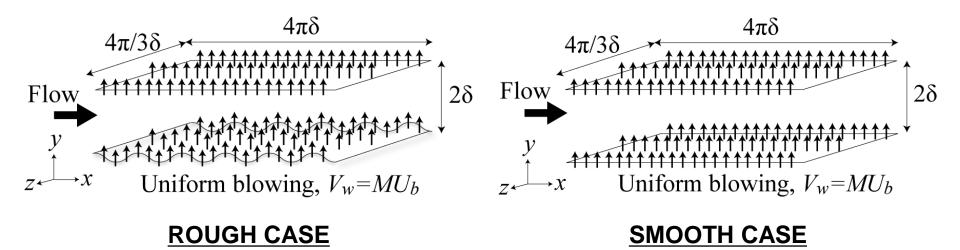
On a boundary layer, White: vortex core, Colors: wall shear stress

# Investigate the interaction between roughness and UB for drag reduction

- DNS of turbulent channel flow
- Focus on drag reduction performance and mechanism

## **Numerical procedure**

- **Based on FD code** (for wall deformation) (Nakanishi et al., *Int. J. Heat Fluid Fl.*, 2012)
- Constant flow rate,  $\operatorname{Re}_b = 2U_b \delta / \nu = 5600$ 
  - so that  $Re_{\tau} \approx 180$  in a plane channel (K.M.M.)
- $\Delta x^+ = 4.4, 0.93 < \Delta y^+ < 6, \Delta z^+ = 5.9$
- UB magnitude: *M* = 0, 0. 001, 0. 005

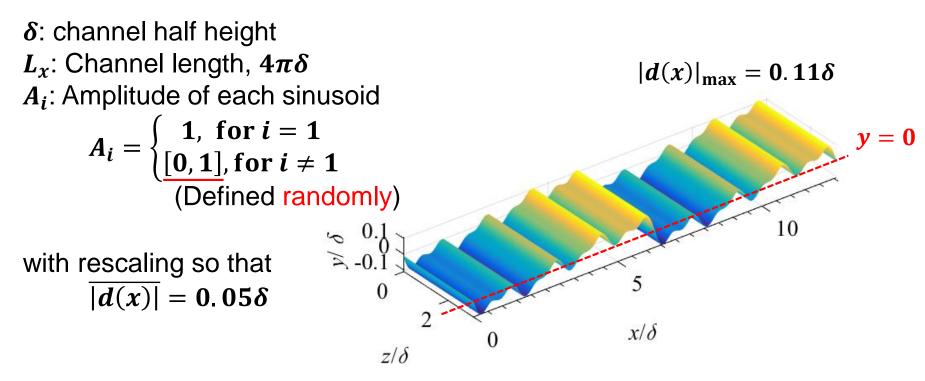


## Model of rough wall

(E. Napoli et al., J. Fluid Mech., 2008)

#### **Roughness displacement**

$$d(x) = \delta \sum_{i=1}^{4} A_i \sin\left(\frac{2^i \pi x}{L_x/2}\right)$$



## **Coordinate transformation**

(S. Kang & H. Choi, Phys. Fluids, 2000)

#### Calculation grids: $\xi_i$ (Cartesian with extra force)

lower/upper wall

DNS/Drag Reduction/Uniform blowing(UB)/Rough wall

0

0.1

0.2

0.3

x

wall

0.5

0.4

grid points allocation

## Post processing

Drag coefficient decomposition for rough case

$$C_{Duf} = \frac{8}{\text{Re}_{b}} \frac{d\bar{u}}{dy}\Big|_{\xi_{2}=2}$$

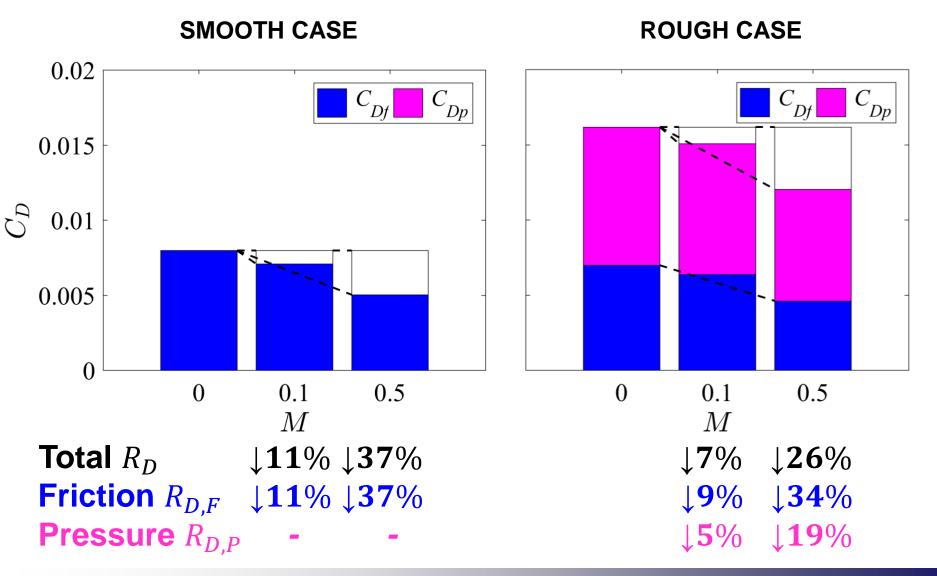
$$C_{Dlf} = \frac{8}{\text{Re}_{b}} \overline{\left(\frac{du}{dy}\Big|_{\xi_{2}=0} + \frac{dv}{dx}\Big|_{\xi_{2}=0}\right)}$$
(Friction component)
$$C_{Dlp} = -16 \frac{dP}{d\xi_{1}} - \left(C_{Dlf} + C_{Duf}\right)$$
(Pressure component)

Drag reduction rate

$$R_{Dl} = \frac{\Delta C_D}{C_{D,M=0}} \times 100 \, [\%]$$

**Only focusing on lower side**, subscript "*l*" omitted hereafter

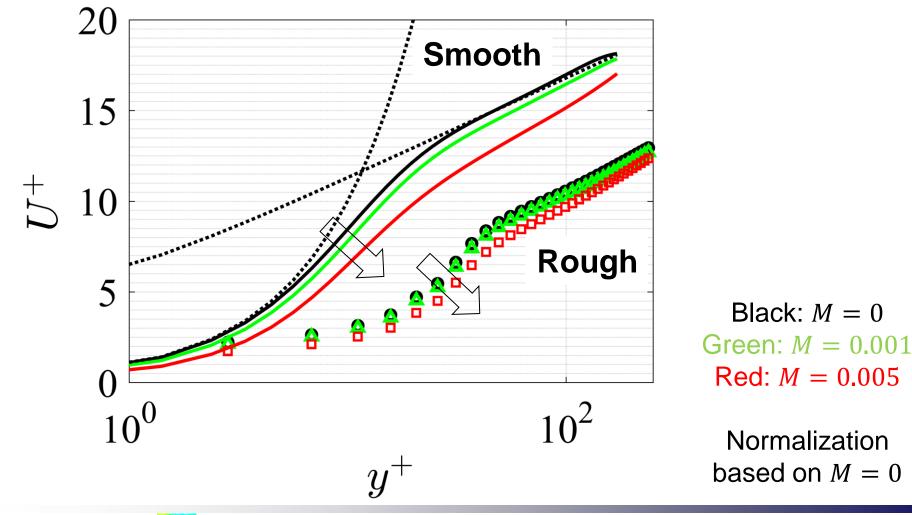
## Drag reduction rate, R<sub>D</sub>



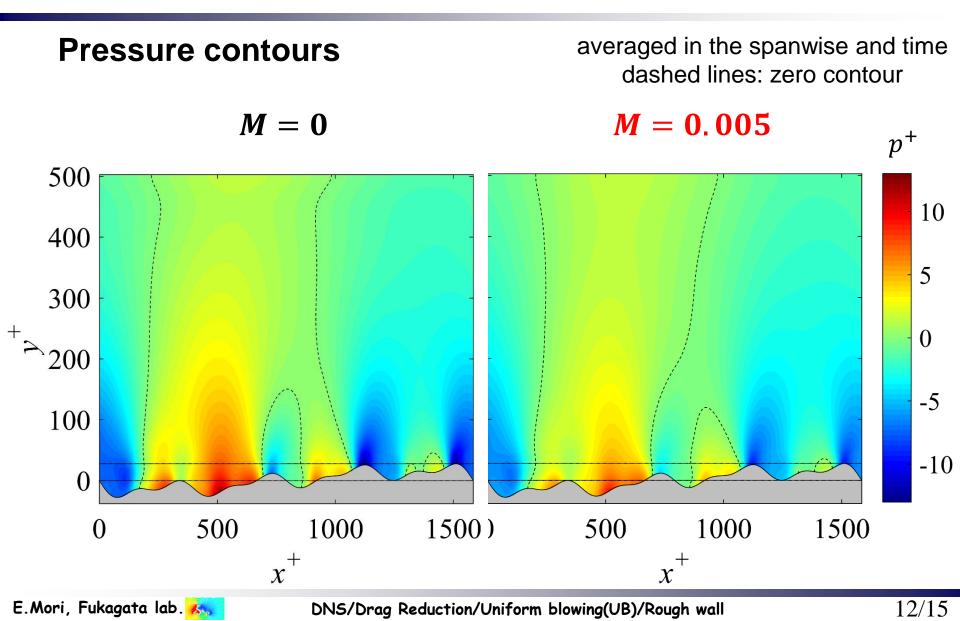
E.Mori, Fukagata lab. 🚲

## How does friction drag decrease?

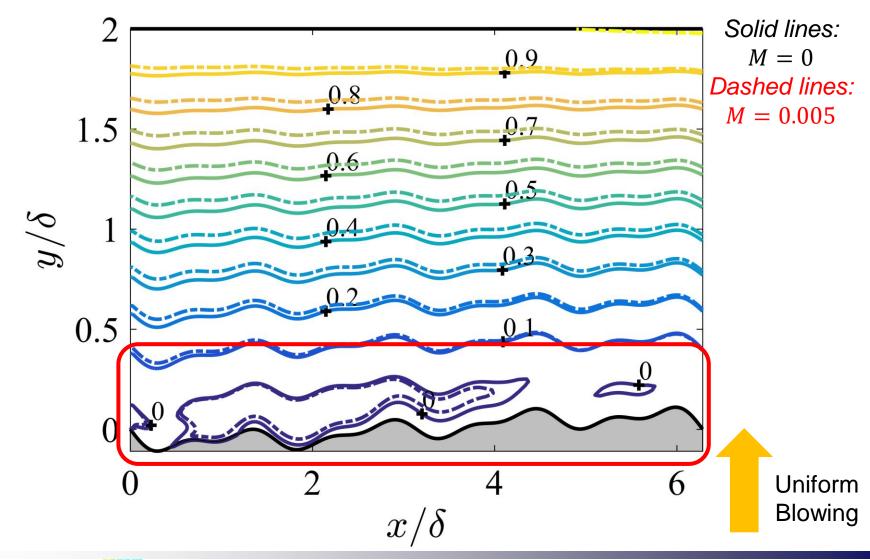
#### Bulk mean streamwise velocity



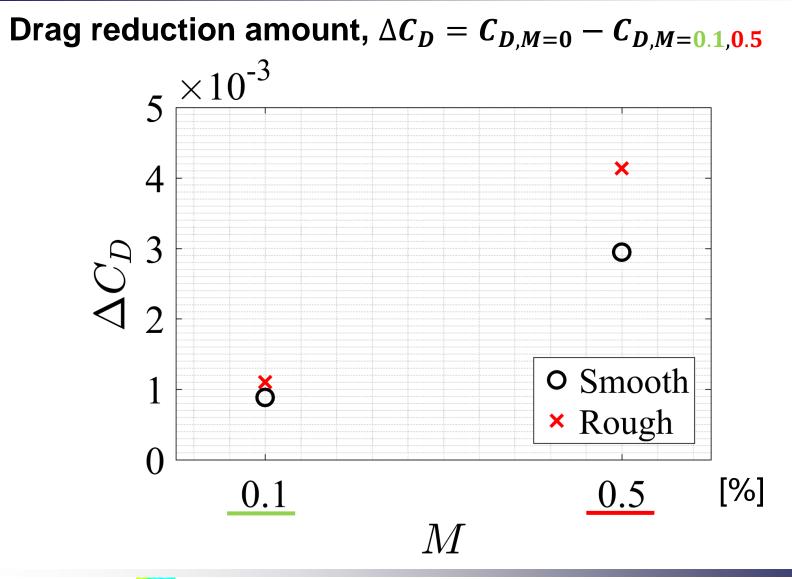
## How does pressure drag decrease?



## **Stream function**



## In practical applications



## **Concluding remarks**

# DNS of turbulent channel flow over a rough wall with UB

- UB is effective over rough walls
  - Lower drag reduction rate (7%, 26% / 11%, 37% in rough / smooth case, with M = 0.001, 0.005)
- Drag reduction mechanism
  - Friction drag by wall-normal convection (=conventional)
  - Pressure drag by prevention of stagnant flow
- Outlook toward practical applications
  - More saving opportunity over rough walls