

Streamwise-traveling waves in a pipe flow: experimentally assessing the turbulent drag reduction

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XX AIDAA Conference – Milano
June 30, 2009

Outline

- 1 Background
- 2 The traveling waves
- 3 Experimental setup
- 4 Results

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Spanwise wall forcing of turbulence

A long story made short

Spanwise forcing may decrease turbulent friction drag:

1985 Bradshaw & Pontikos 1985: **sudden** spanwise pressure gradient

1992 Jung et al. 1992: **harmonic** spanwise wall oscillation

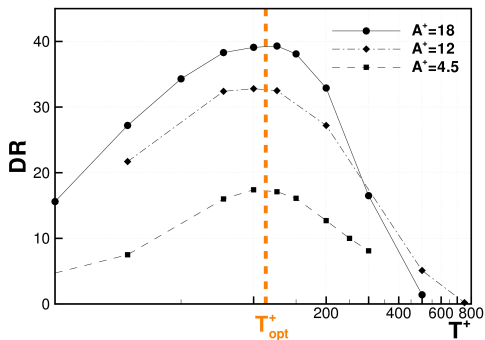
1993- many papers on the oscillating-wall technique

Spanwise wall oscillation: the essentials

Quadrio & Ricco, JFM 04

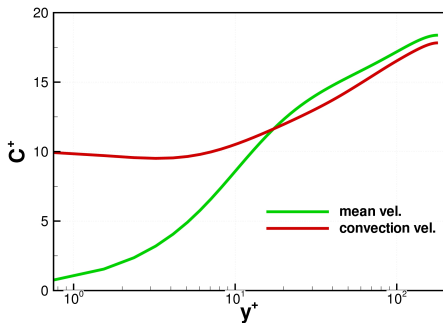
$$w(x, y = 0, z, t) = A \sin(\omega t)$$

- Large reductions of turbulent friction
- Basic mechanism still elusive
- Does an optimum period T_{opt} exist?
- Unpractical



An important concept: the convection velocity

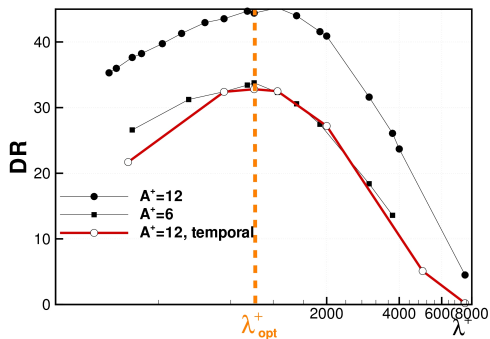
- Turbulent fluctuations **at the wall** possess a convection velocity
- Known concept (Kreplin & Eckelmann) in the '70
- Re-discovered (!) by Kim & Hussain '93
- Re-re-discovered (!!) by Quadrio & Luchini '03



The oscillating wall made stationary

$$w(x, y = 0, z, t) = A \sin(\kappa x)$$

- Convection allows translating the oscillation into a **steady forcing**
- Existence of an **optimal wavelength**
 $\lambda_{opt} = U_w T_{opt}$
- Easily implemented as a **passive device** (sinusoidal riblets)



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The traveling waves: a natural extension

Purely temporal forcing

The **oscillating** wall:

$$w = A \sin(\omega t)$$

- Infinite phase speed

Purely spatial forcing

The **steady** waves:

$$w = A \sin(\kappa x)$$

- Zero phase speed

Combined space-time forcing

The **traveling** waves:

$$w = A \sin(\kappa x - \omega t)$$

- Finite phase speed $c = \omega/\kappa$

First results: a DNS study

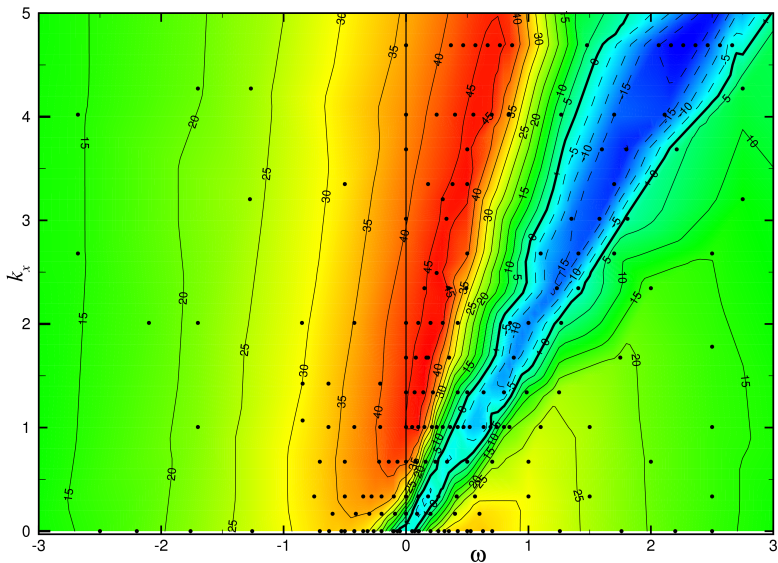
Quadrio et al., JFM 2009

- DNS pseudo-spectral code
- Powerful system with 268 dual-core Opteron CPUs, 280GB RAM, 40TB disk space
- Turbulent **plane channel flow** at $Re_\tau = 200$
- Approx. **4 centuries** of CPU time (or 500MWh of power)



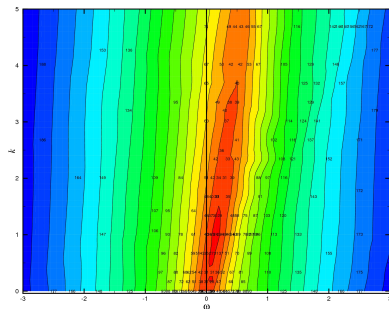
Unexpected results!

Waves may yield both DR and DI



How much power to generate the waves?

- Power $\sim w \partial w / \partial y|_{y=0}$
- Upper bound to energetic cost
- Similar to drag reduction map!
- Ratio of energy save to cost up to **30:1**
- Up to 25% net energy save



Motivation for a laboratory experiment

Devise a **proof-of-principle** experiment to:

- **confirm** DR and DI
- improve our understanding of the traveling waves
- explore further the parameter space (Re , A)

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Our experimental setup

The main design choices

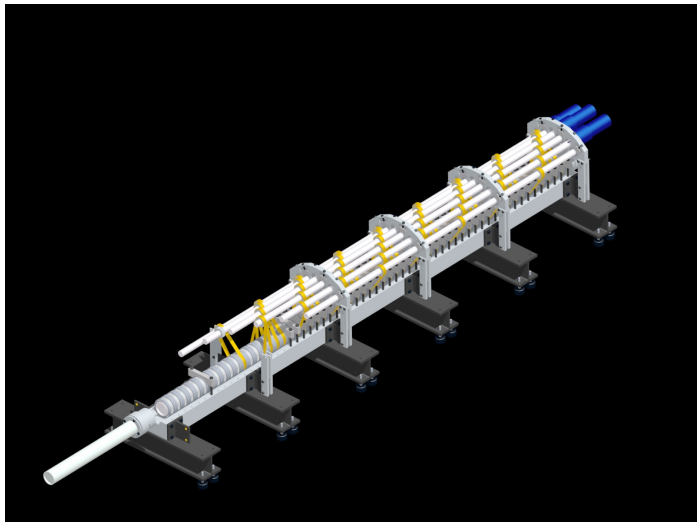
- Geometry of **cylindrical pipe**: naturally periodic in spanwise (azimuthal) direction
- Friction is measured through pressure drop
- Spanwise velocity at the wall is achieved by moving the wall
- Temporal variation is achieved by unsteady control of the wall velocity
- Spatial variation is achieved by **slicing the pipe into** thin, independently-movable **axial segments**

Notable difficulties

- Low-budget experiment
- Small pressure drop
- Water

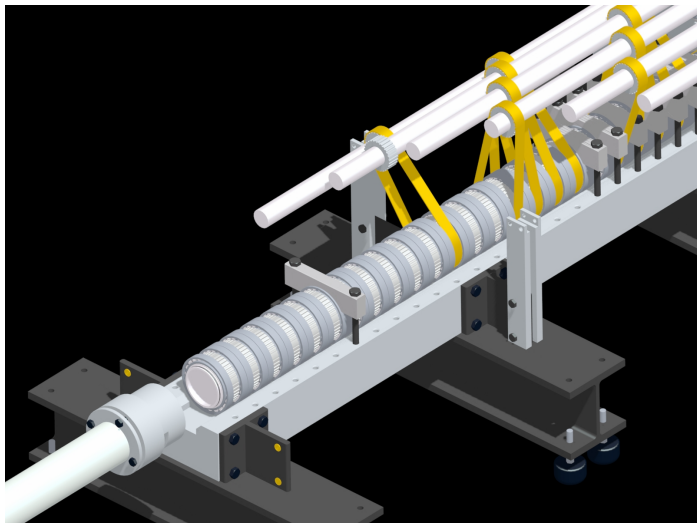
The pipe

A closed-circuit water pipe



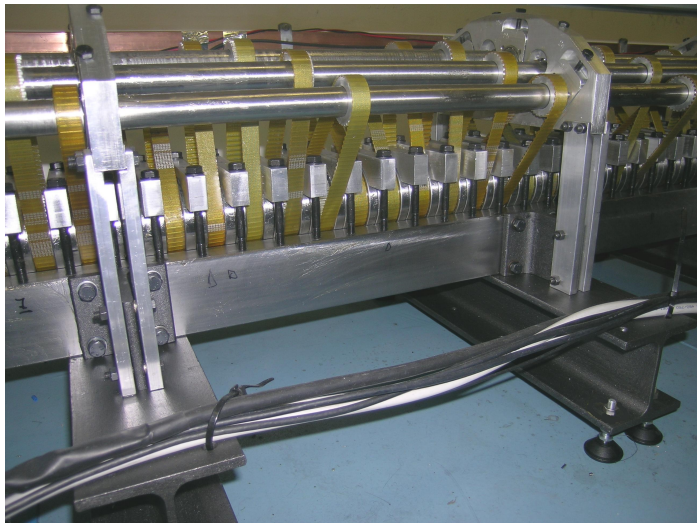
The rotating segments

60 slabs with 6 independent motors



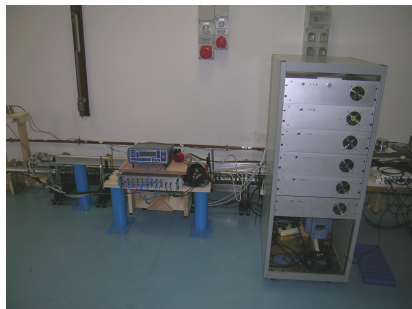
The transmission system

Shafts, belts and rotating segments

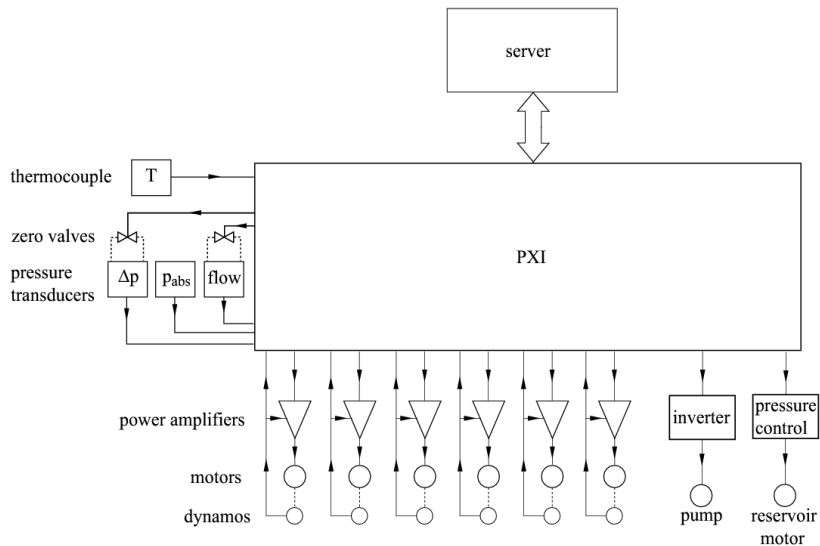


The control system

- Motion of the slabs is feedback-controlled
- Dynamometric sensors to feed back angular speed
- Fully automated test management



Schematic



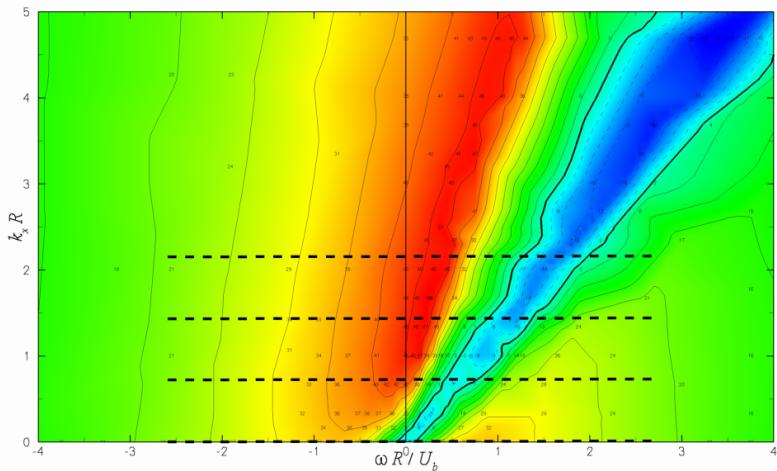
Flow parameters and procedure

- Working fluid is water
- $U_b = 0.9$ m/s
- $Re = 4900$
- $Re_\tau = 180$
- System degassed after filling
- **Temperature** is continuously monitored
- Flow rate is continuously monitored
- Re is adjusted at every measurement point by changing U_b

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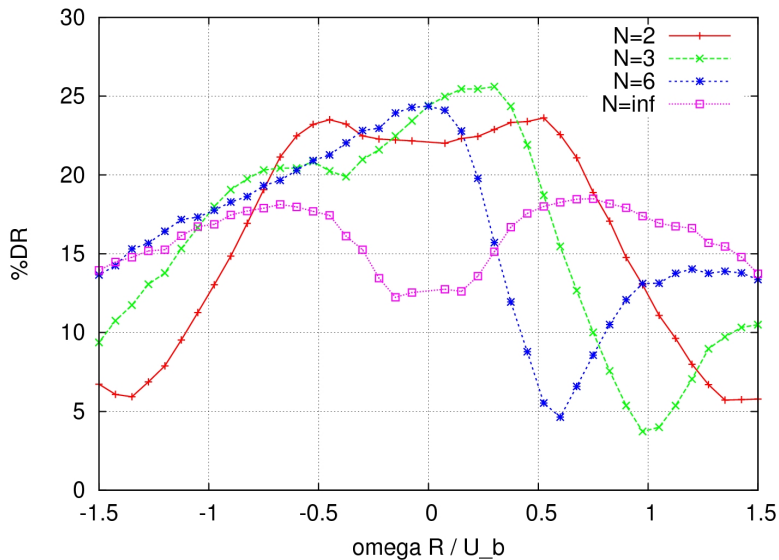
Experimental conditions



SHOW MOVIE HERE?

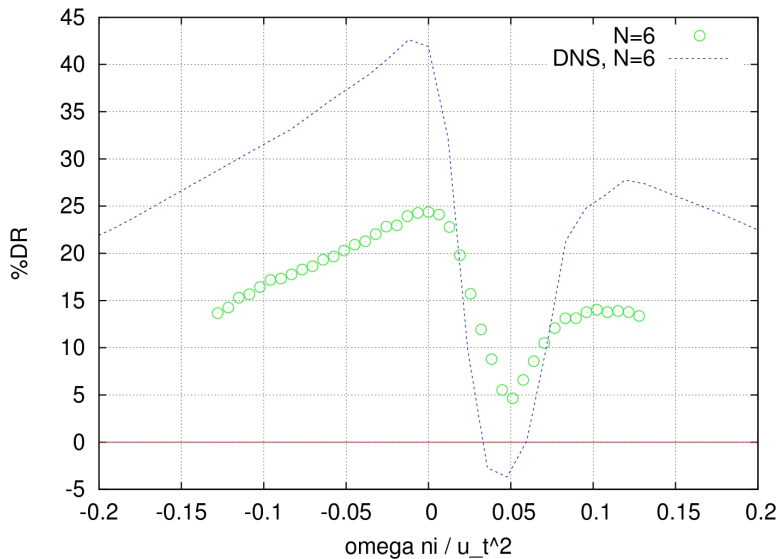
Drag changes

Bulk units



Comparison with DNS & plane channel

Inner units



Discussion

Quantitative disagreement between DNS and experiment

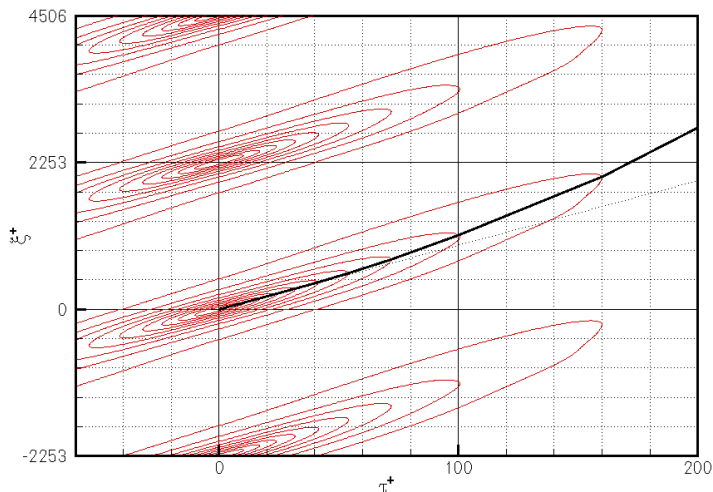
- Spatial transient?
- Cylindrical vs planar geometry?
- Difference in GSL
- Difference (small) in Re and A

Conclusions

- DR is **confirmed**
- 26% DR is one of the largest ever measured
- Quantitative uncertainty
- Demonstrated existence of T_{opt} for oscillating pipe

Understanding the physics

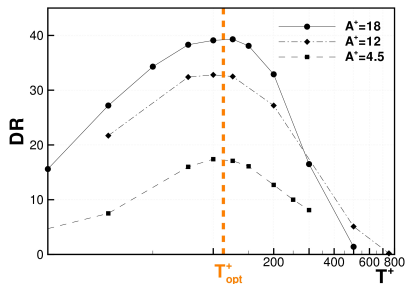
The lifetime T_ℓ of turbulent structures



Unsteadiness in the convecting reference frame

Oscillating wall

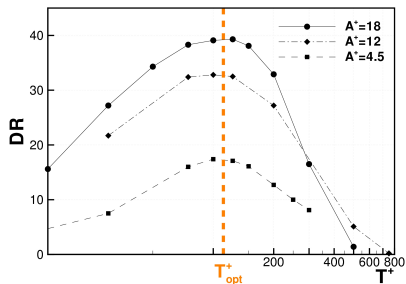
- Forcing on a timescale $\gg T_\ell$ does not yield DR
- Timescale: oscillation period T



Unsteadiness in the convecting reference frame

Oscillating wall

- Forcing on a timescale $\gg T_\ell$ does not yield DR
- Timescale: oscillation period T



Traveling waves

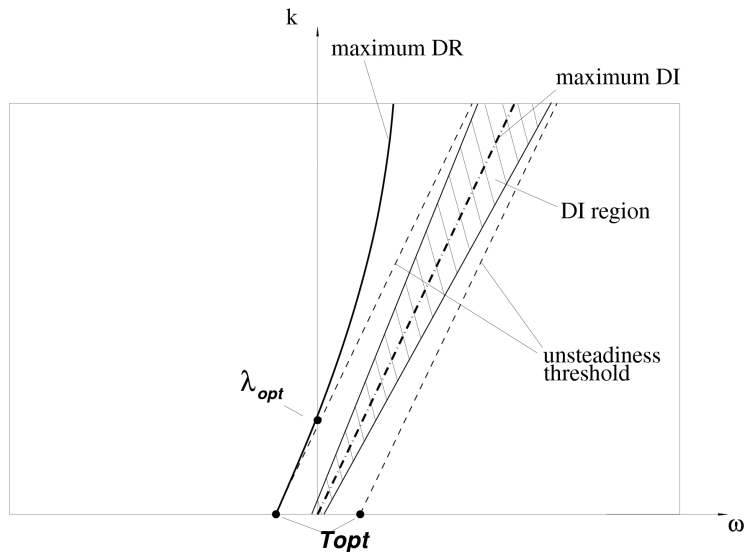
- Forcing on a timescale $\gg T_\ell$ does not yield DR
- Timescale: oscillation period \mathcal{T} as seen in a **convecting reference frame**

$$\mathcal{T} = \frac{\lambda_x}{U_w - c}$$

- U_w : convection velocity at the wall
- $c = \omega/\kappa$: phase speed

How spanwise forcing really works

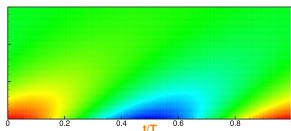
Quadrio et al., JFM 2009



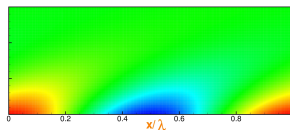
One step back

Extending the laminar Stokes solution

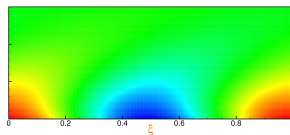
- Laminar flow
- Transverse, alternating boundary layer
- Qualitative similarity



$$w(y, t)$$



$$w(y, x)$$

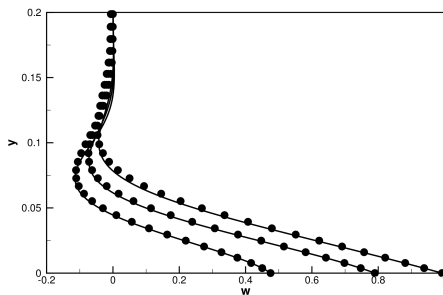


$$w(y, x - ct)$$

The generalized Stokes layer

An analytical approximate solution

$$w(x, y, t) = A \Re \left\{ C e^{2\pi i(x-ct)/\lambda_x} \text{Ai} \left[e^{\pi i/6} \left(\frac{2\pi u_{y,0}}{\lambda_x \nu} \right)^{1/3} \left(y - \frac{c}{u_{y,0}} \right) \right] \right\}$$



- $\delta_{GSL} \ll h$
- Neglect streamwise viscous diffusion
- Threshold velocity to discriminate flow regimes

Using the GLS solution

Thickness of the GLS

